

# GREAT DESIGNS IN **STEEL**™



**GDIS Technical Webinar Series:**

**A novel approach for precise calibration  
of force-based RSW failure criteria  
implemented in LS-DYNA**

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**December 10, 2024**

# Presentation outline

## Introduction

- 3G-AHSS
- RSW Process
- RSW Failure Prediction
- RSW Component Tests
- Research Motivation

## Task 1

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- Weldability Lobes
- Failure Surfaces

## Task 2

- Shear-tensile Loading via KS-II Tests
- Failure Loci Assessment in Shear-tensile Mode

## Task 3

- Tensile-bending Loading via Coach Peel Tests
- Failure Loci Assessment in Tensile-bending Mode

## Task 4

- Tensile-bending Loading of Weld-groups (Component Tests)
- Mechanical Properties
- Failure Behavior
- LS-DYNA Simulations

## Conclusions

- Main Conclusions

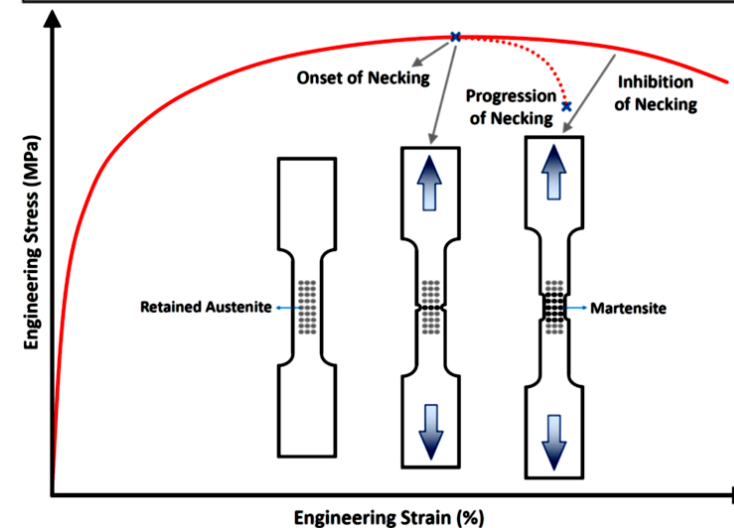
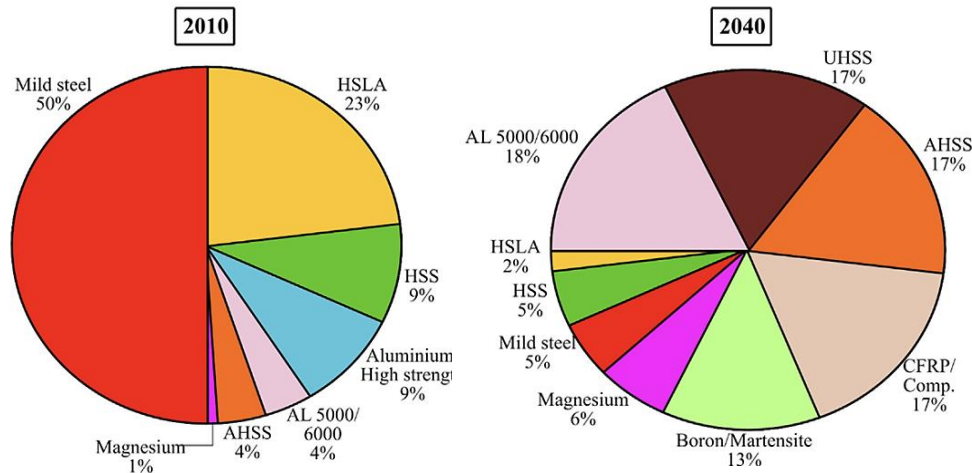
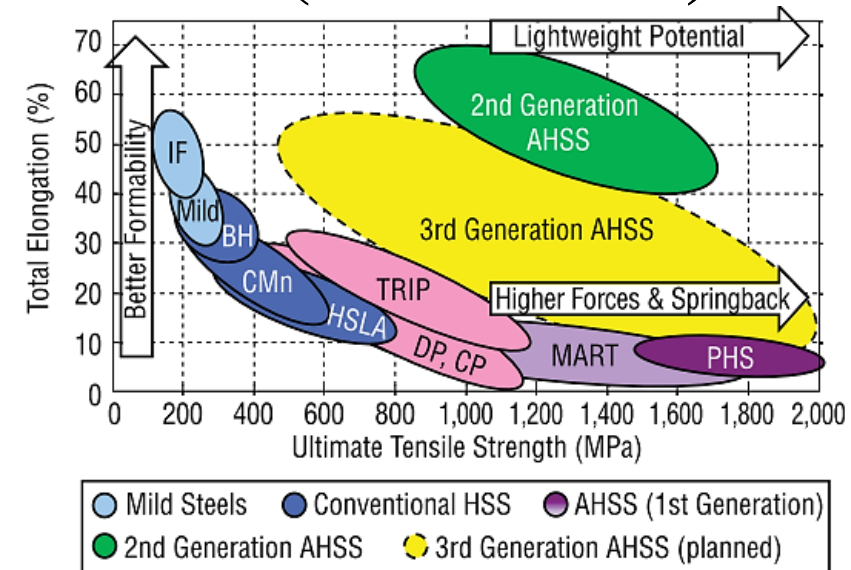
# Third Generation Advanced High Strength Steels (3G-AHSS)

Objectives in Automotive Industry

Improving Fuel Efficiency

Enhancing Safety

AHSS

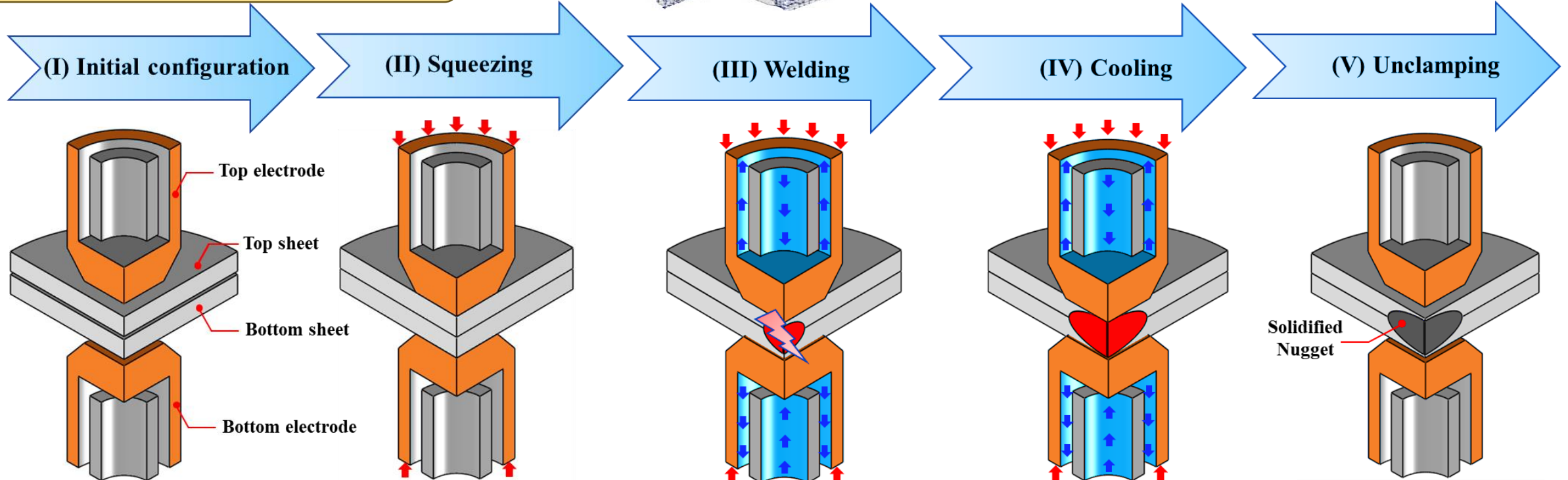
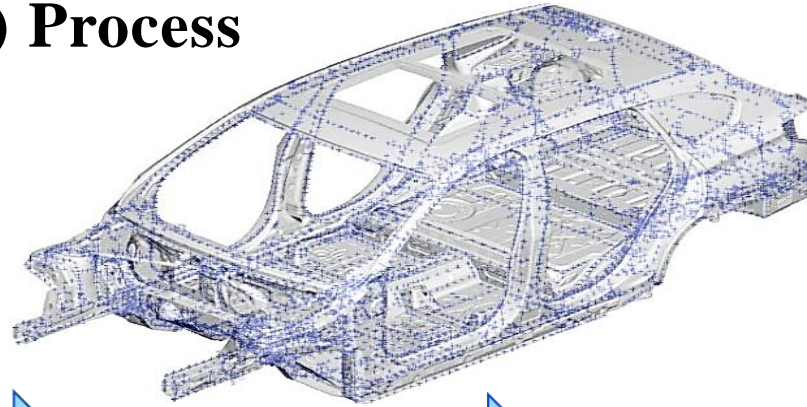


[1] Giampieri et al, *Applied Energy* (2020)  
 [2] Zhao et al, *Progress in Materials Science* (2018)  
 [3] Soleimani et al, *Materials Science and Engineering* (2020)

# Resistance Spot Welding (RSW) Process

RSW is the most common joining technique in the automotive industry

5000 Spot Welds in a typical body-in-white on average

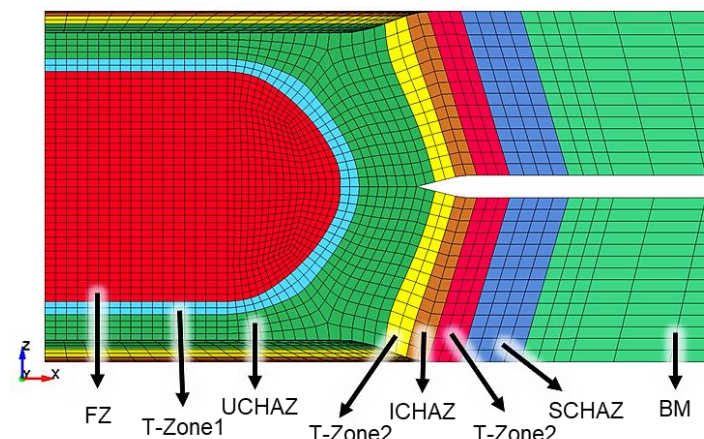
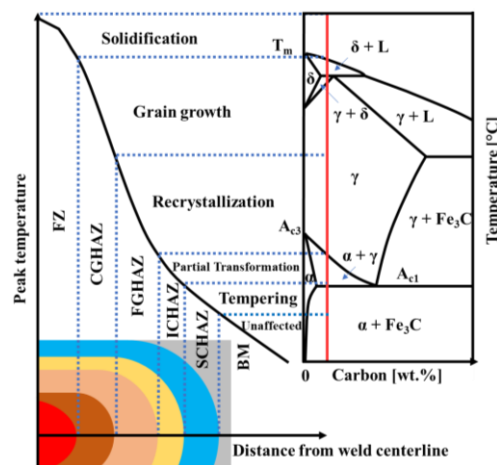
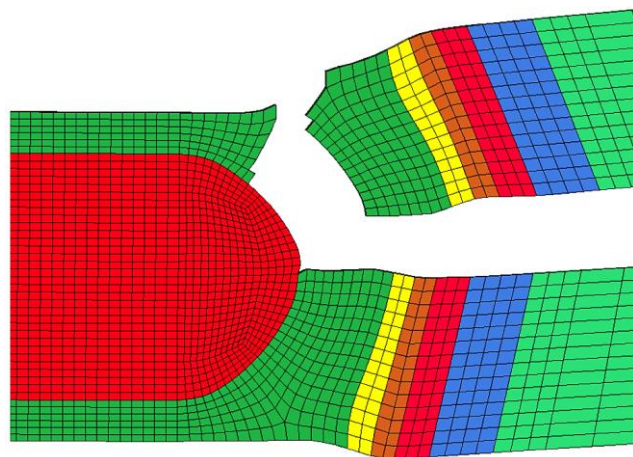


Predicting failure of spot welds is a priority for the automotive industry

[4] Zerohourparts.com (2024)

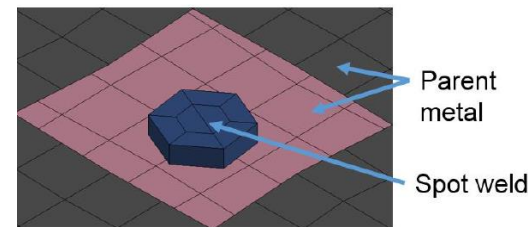
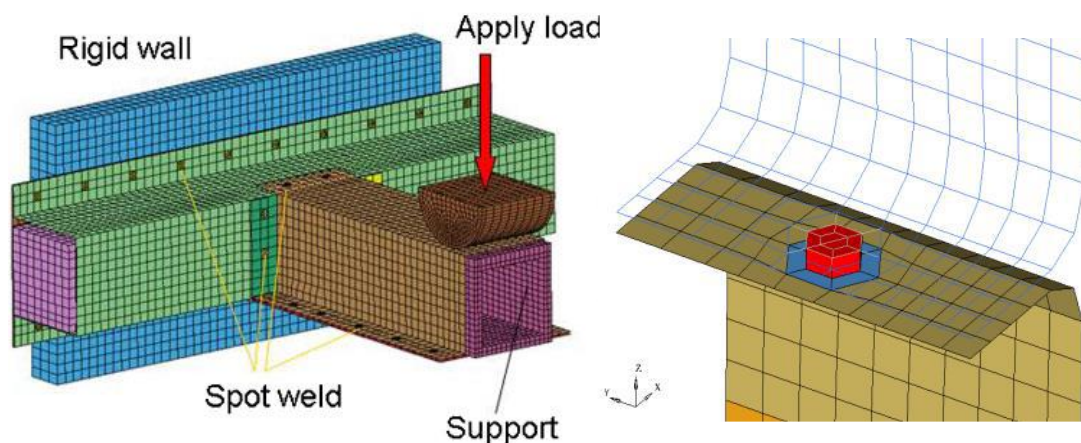
# RSW Failure Prediction:

## 1. Mesoscale models (fine mesh + constitutive properties of sub-zones)

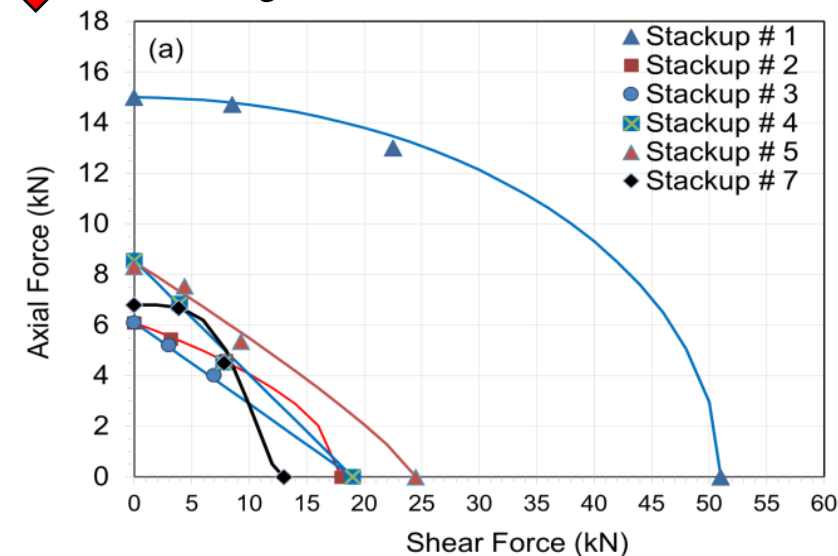


- ↑ Informative regarding mechanics of fracture
- ↓ Computationally demanding, Requires fracture characterization of sub-zones

## 2. Force-based models (coarse mesh + interpolated failure loci from mechanical tests)



- ↑ Well-suited for industrial applications
- ↓ Limited insight into mechanics of failure



[5] Mohamadizadeh Ph.D. Thesis (2020)

[6] Ghassemi-Armaki et al. *Int J Mech Sci* (2011)

[7] Midawi et al. *Forces in Mech* (2022)

# Load-based RSW Failure Criteria:

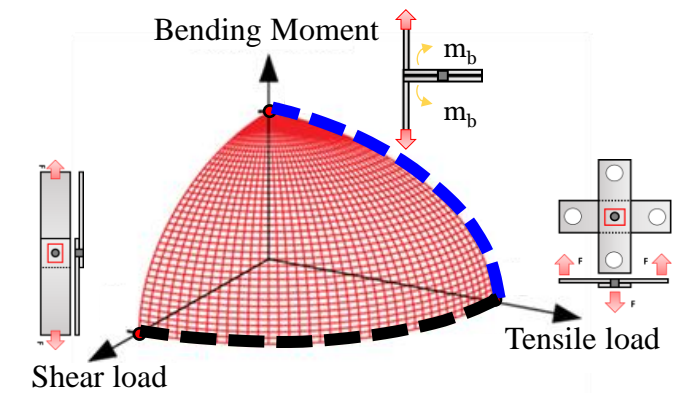
Failure equation in  
LS-DYNA \*MAT\_100\_DA  
material model:

$$f^{Seeger} = \left[ \frac{f_s}{S_S(\dot{\epsilon}_{eff})} \right]^a + \left[ \frac{f_n}{S_N(\dot{\epsilon}_{eff})} \right]^b + \left[ \frac{m_b}{S_B(\dot{\epsilon}_{eff})} \right]^c \geq 1$$

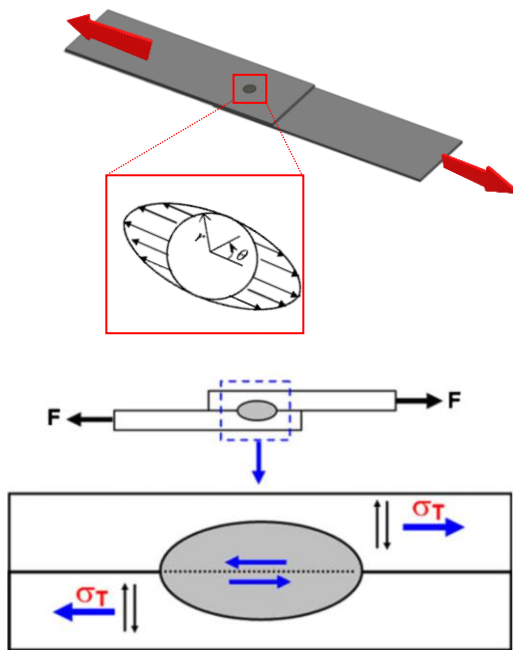
Shear strength

Tensile strength

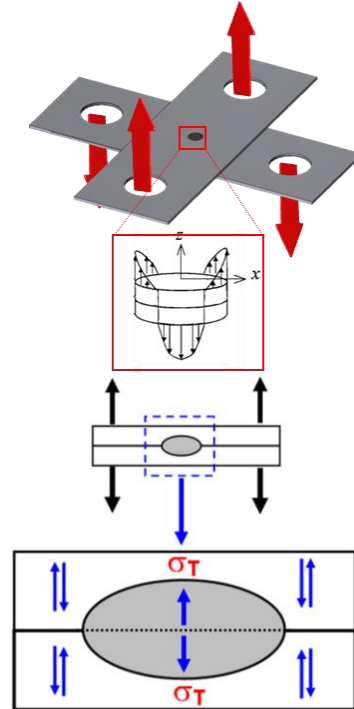
Bending strength



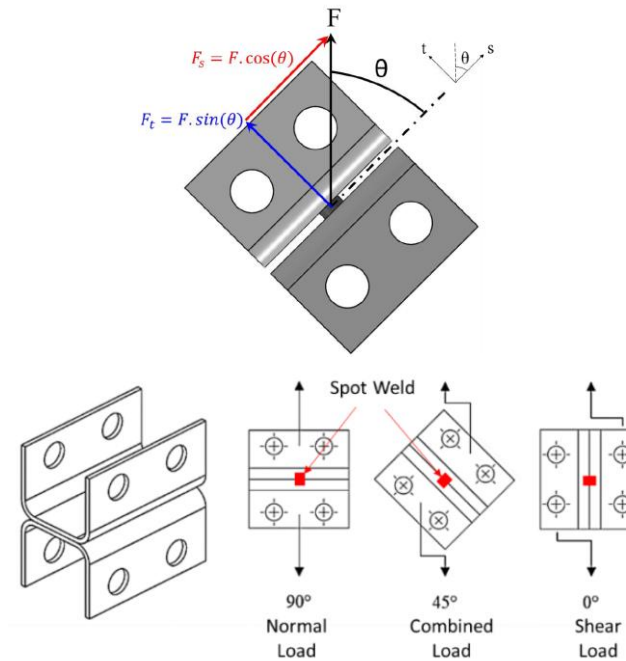
## Lap Shear Test



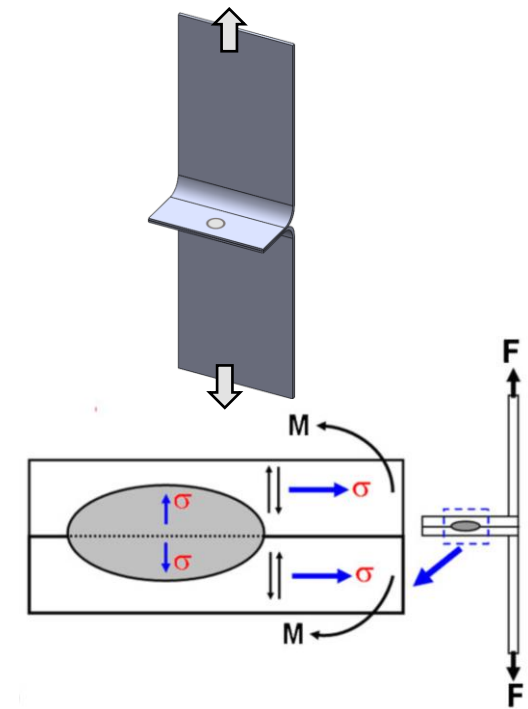
## Cross-tension Test



## KS-II Test

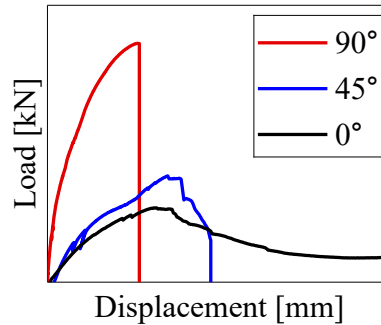
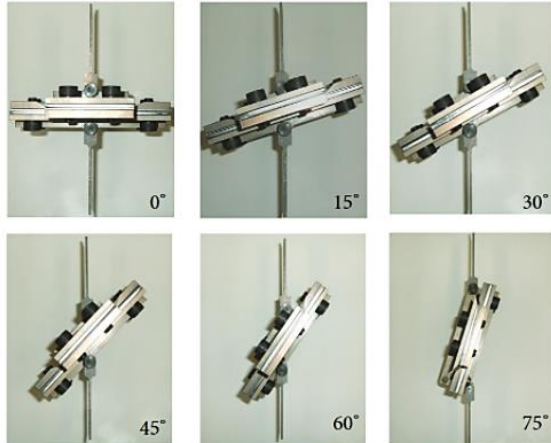
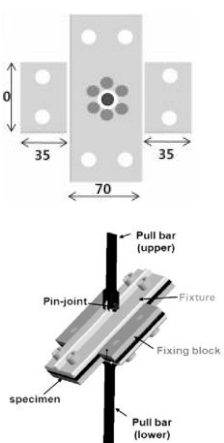


## Coach Peel Test



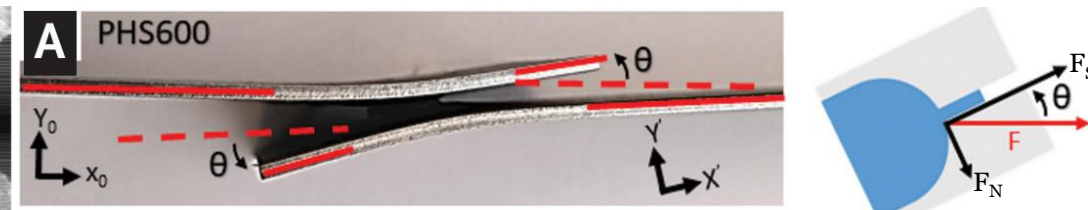
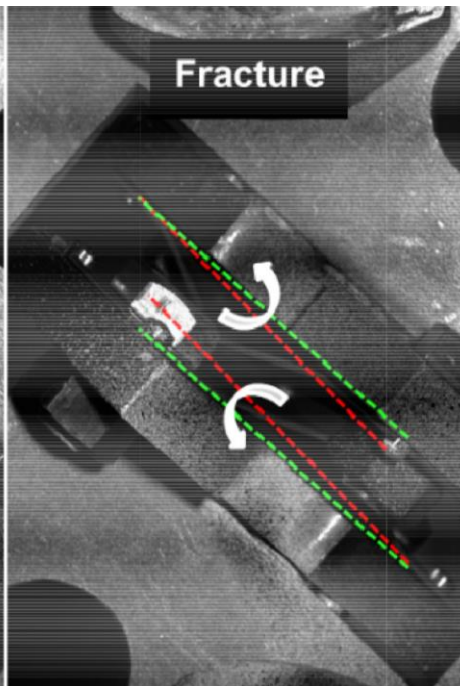
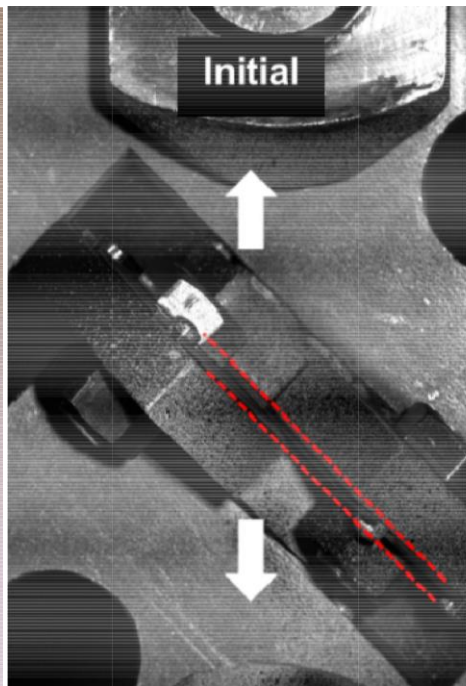
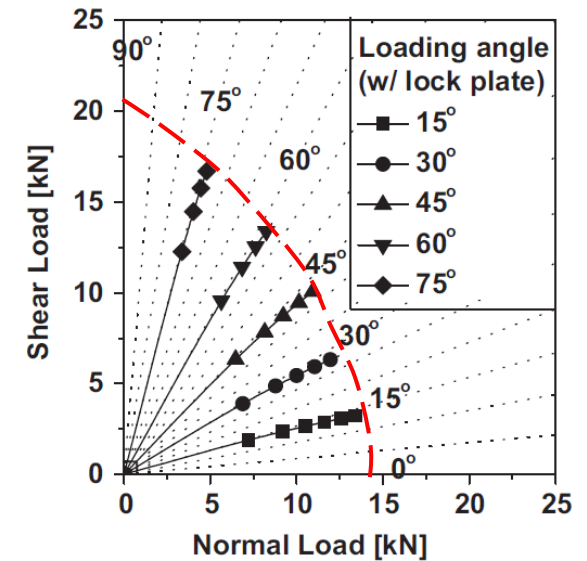
- [8] Song JH, Huh H. *Int J Mech Sci* (2011)  
 [9] Pouranvari et al, *J Mater Eng Perform* (2012)  
 [10] Seeger, F., et al. *4th LS-DYNA User Forum*, (2008)

# Combined Shear-tensile Loading of Spot Welds



$$F_{Shear} = F \cdot \cos(\theta)$$

$$F_{Tensile} = F \cdot \sin(\theta)$$

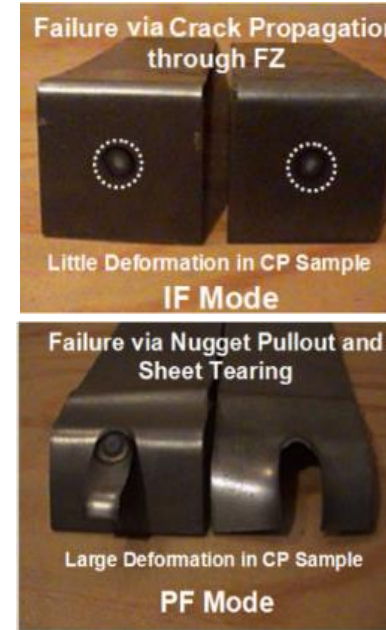
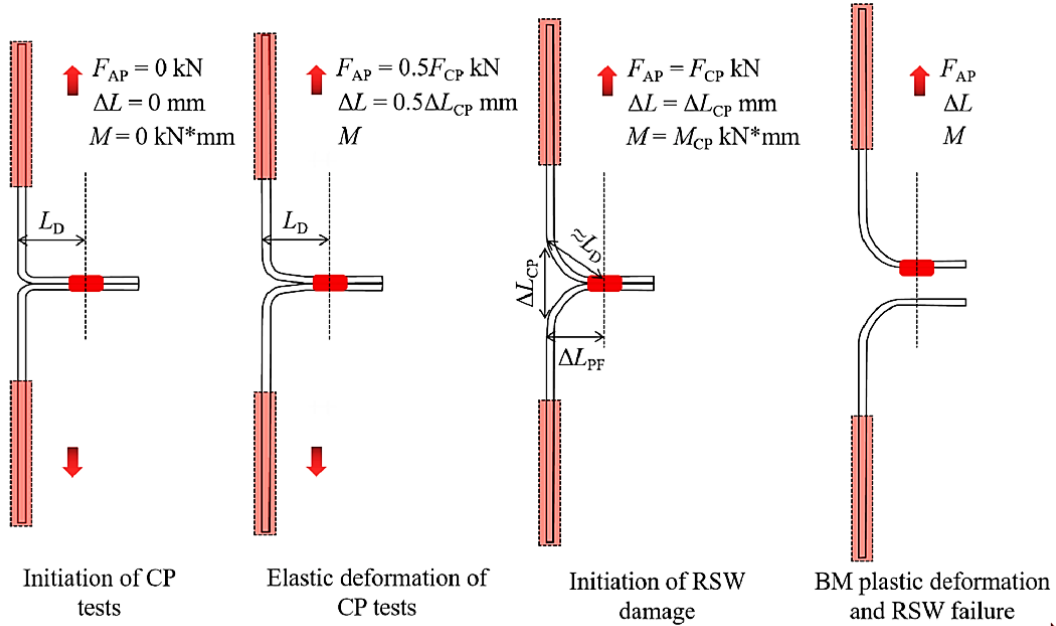


## Deficiencies in shear/tensile loading of spot welds:

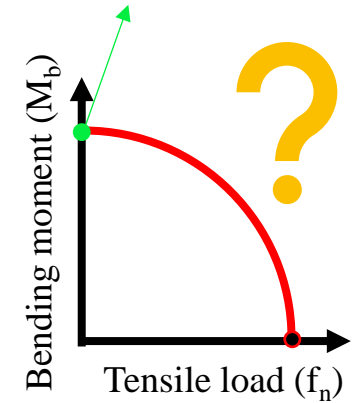
- Quantification of coupon slippage and rotation
- Inaccurate critical shear and tensile load components

# Combined Tensile-bending Loading of Spot Welds

$$f_{Seeger} = \left[ \frac{f_n}{S_N(\dot{\epsilon}_{eff})} \right]^a + \left[ \frac{f_s}{S_S(\dot{\epsilon}_{eff})} \right]^b + \left[ \frac{m_b}{S_B(\dot{\epsilon}_{eff})} \right]^c \geq 1$$



$$S_B = \frac{F_{Max CP} \times d}{\sqrt{1 - \left( \frac{F_{Max CP}}{F_{Max KS-II 90^\circ}} \right)^2}}$$



## Deficiencies in tensile/bending loading of spot welds:

- No assessment of convex failure loci assumption
- Idealized deformation of the coach peel coupon
- Unknown evolution of bending moment and its value at the onset of failure

$$M_{CP} = F_{CP} \times \Delta L_{PF}$$

Bending moment      Load      Moment arm

[15] Xu et al. *Eng Frac Mech* (2023)

[17] Arumugam et al. *J Mech Eng* (2022)

[16] Pouranvari et al. *Iron Steel Proc* (2012)

[18] Shashikumar et al. *Metals* (2023)

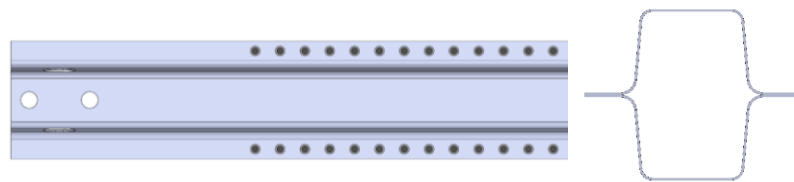
[19] Wang et al. *Int J Crash* (2018)

# Tensile-bending Loading of Groups of Spot Welds

Purpose for Performing Weld-group Tests

Characterization of sequential failure between welds and validate models

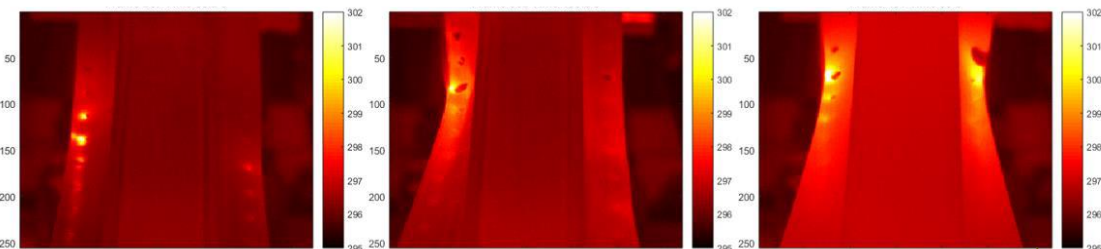
The Caiman Test



Fully Quenched Flange

400 °C Tailored Flange

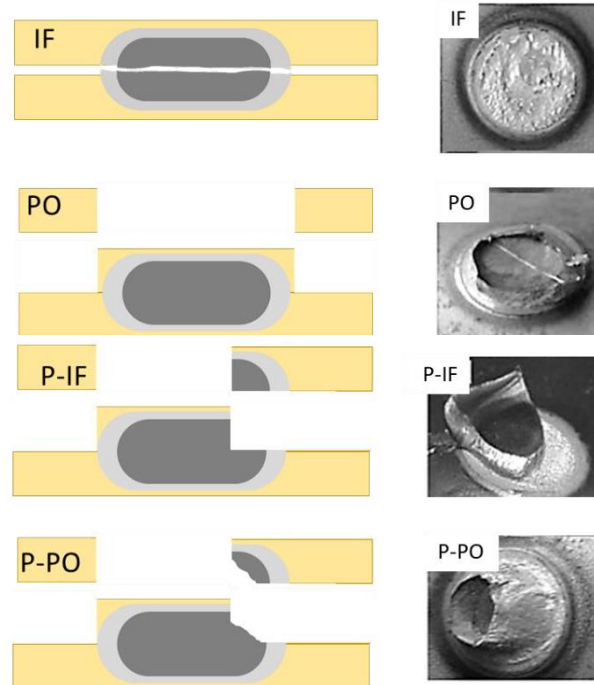
700 °C Tailored Flange



Increase in flange ductility

Higher flange deformation

Improved Energy Absorption



**Deficiencies in tensile-bending loading of spot weld groups:**

- How does failure mode affect the mechanical performance of weld groups?
- What are the factors that dictate energy absorption capability?

[20] Tolton et al, *Master's Thesis* (2021)

[21] O'Keeffe et al, *Master's Thesis* (2018)

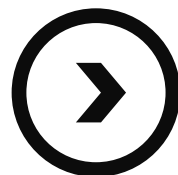
[22] Mohamadizadeh et al, *Exp Mech* (2022)

# Motivation Based on the Identified Gaps Within the Literature:



## Process Optimization

Selecting a Suitable Welding Current That Produces Expulsion-free Joints with the Highest Load Bearing Capacity and Energy Absorption Capability



## Mechanical Properties of Joints Under Shear/Tensile Loading

Assess the Validity of Seeger's Proposed Failure Loci in Shear-tensile Loading Conditions via KS-II Tests



## Mechanical Properties of Joints Under Tensile/Bending Loading

Assess the Validity of Seeger's Proposed Failure Loci in Tensile-bending Loading Conditions via Coach Peel Tests



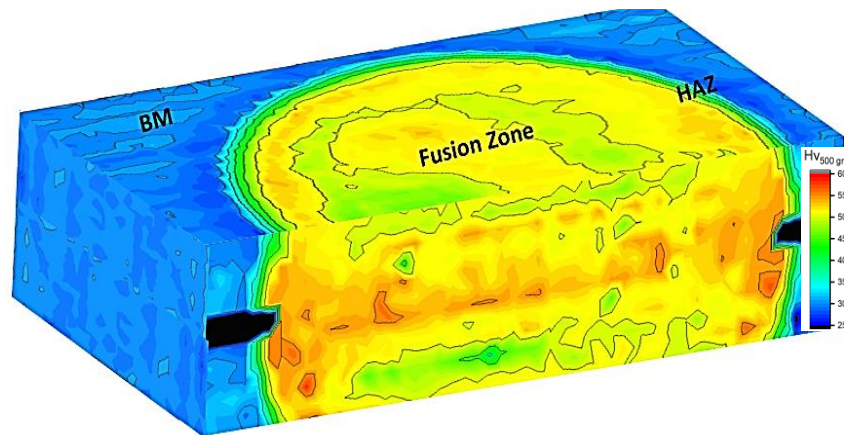
## Mechanical Properties of Groups of Joints Under Tensile/Bending Loading

Critical Factors Influencing Mechanical Performance of The Caiman Mode I Components

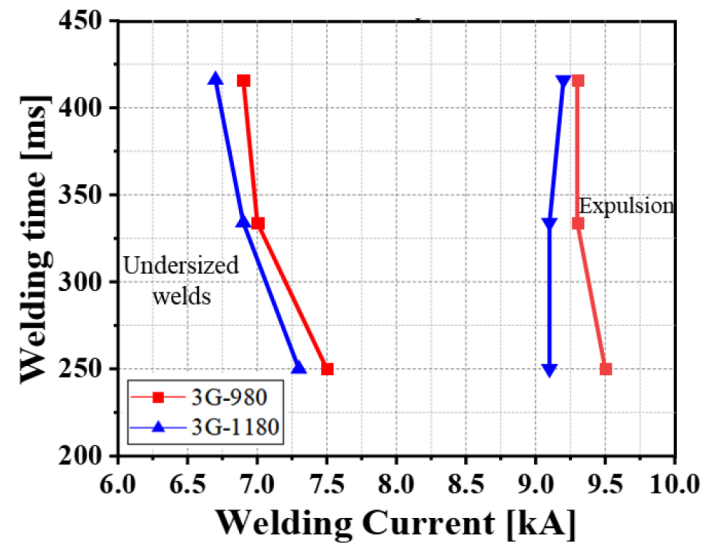
# Task 1

## RSW Process Optimization and Seeger's Nominal Failure Surfaces

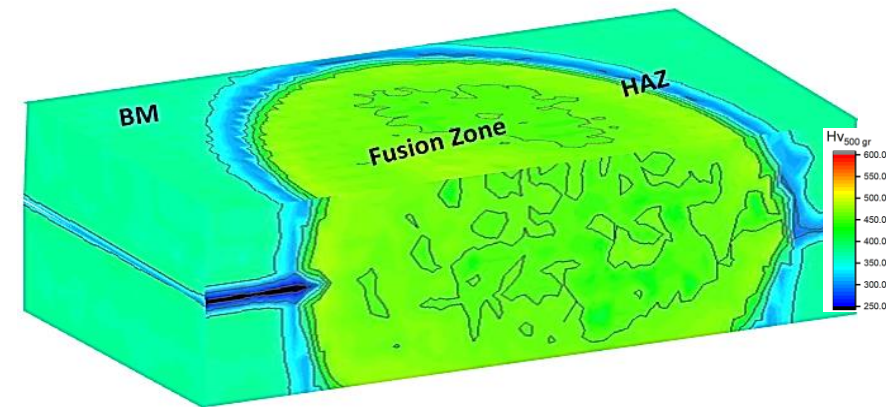
3G-980 Optimized Spot Weld  
(Nugget size of 7 mm)



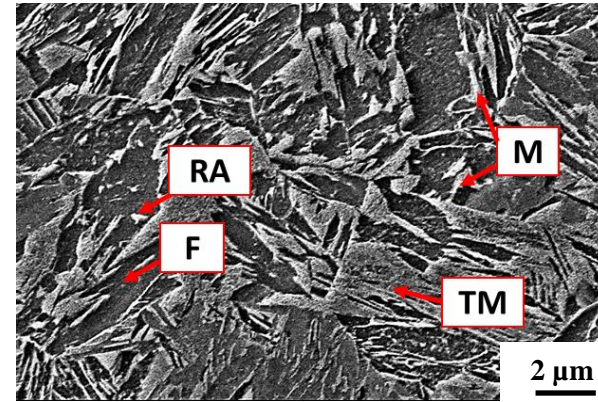
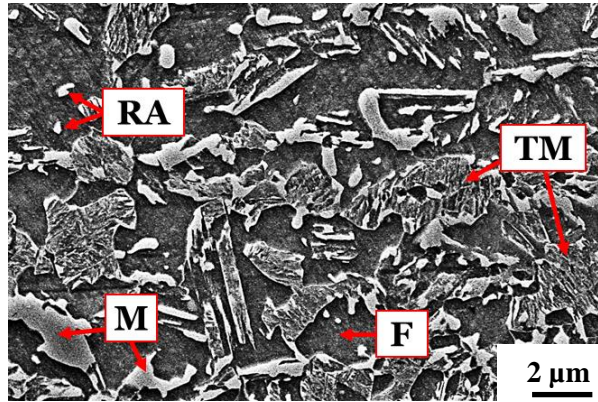
Weldability Lobes



3G-1180 Optimized Spot Weld  
(Nugget size of 7 mm)

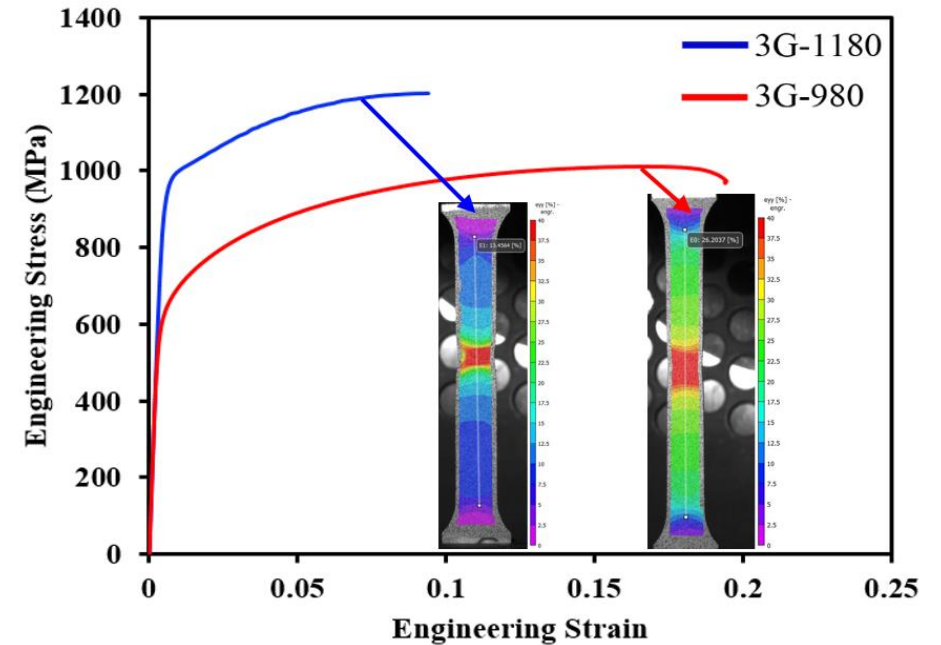


# Investigated Materials



3G-980 (uncoated)  
1.4 mm thickness  
Base material hardness:  
 $300 \pm 6$  HV

3G-1180 (uncoated)  
1.4 mm thickness  
Base material hardness:  
 $380 \pm 5$  HV

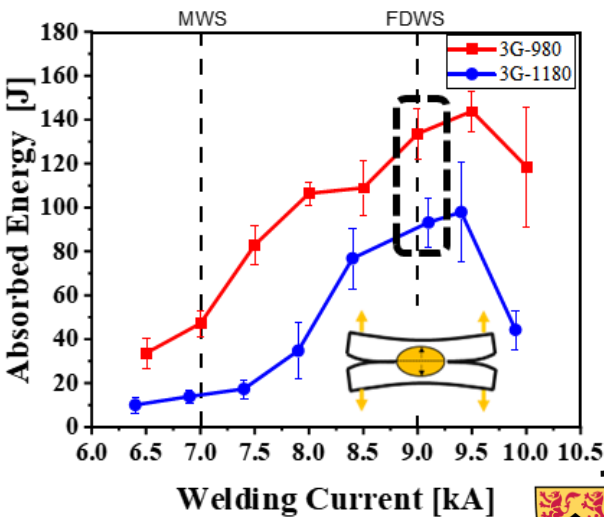
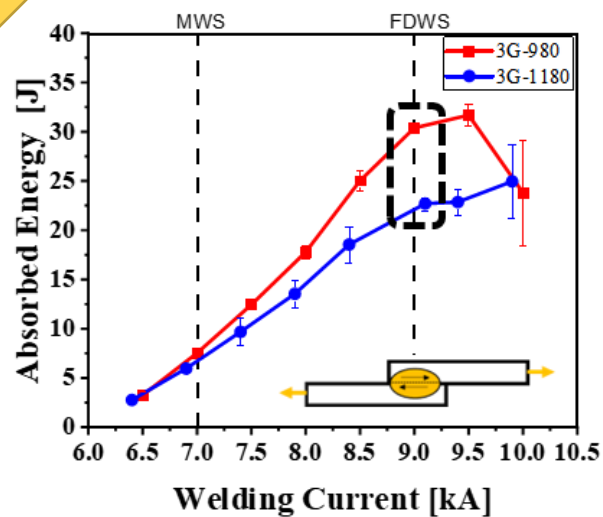
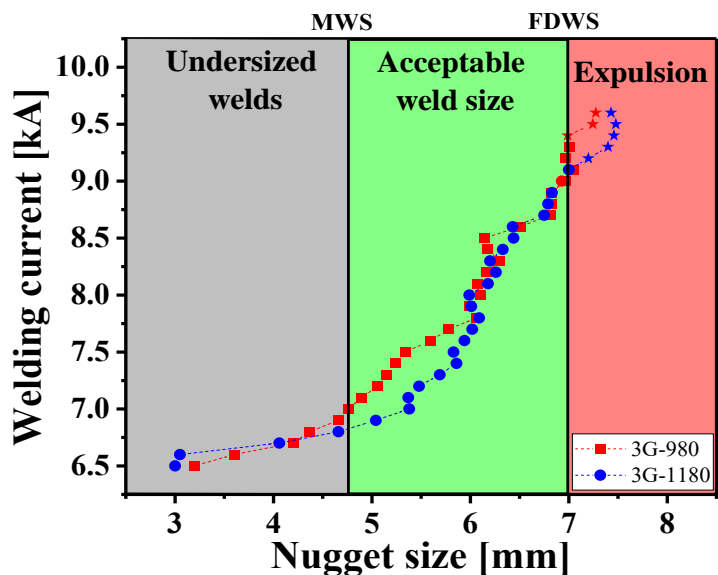
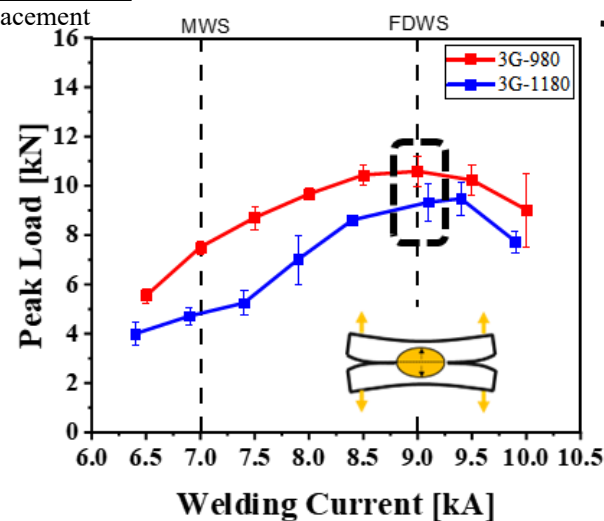
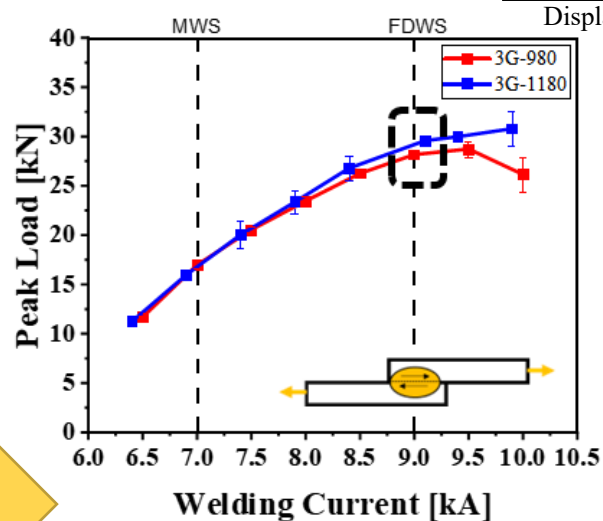
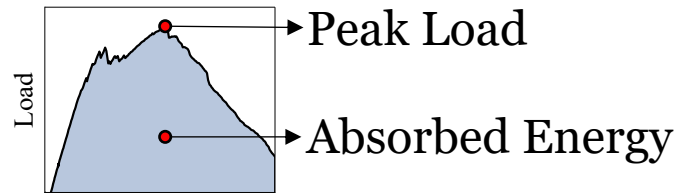
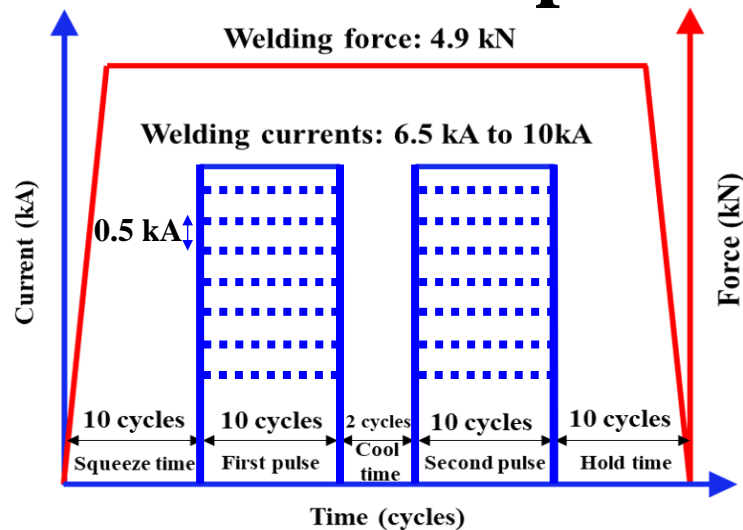


Material	C	Mn	Si	Cr+Mo+V	Cu+Ni	C <sub>eq</sub>
3G-980	0.21	2.11	1.49	0.02	<0.01	0.64
3G-1180	0.17	2.52	1.59	0.28	<0.02	0.70

Material	Sheet Thickness [mm]	YS [MPa]	UTS [MPa]	TE [%]	UE [%]
3G-980	1.4	$605 \pm 7.78$	$1002 \pm 8.29$	$19.89 \pm 0.50$	$16.8 \pm 0.55$
3G-1180	1.4	$967 \pm 7.05$	$1181 \pm 19.02$	$11.83 \pm 0.6$	$8.1 \pm 0.35$

YS, yield strength; UTS, ultimate tensile strength; TE, total elongation; UE, uniform elongation; 0.2% offset method was used to calculate YS

# RSW Process Optimization



Optimized Welding Currents Based on AWS recommendations

9.1 kA for 3G-980

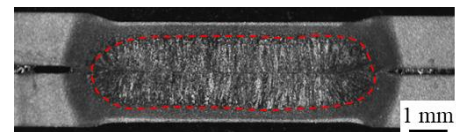
9.0 kA for 3G-1180

FDWS: Electrode Face Diameter Weld Size ( $\approx 7$  mm)

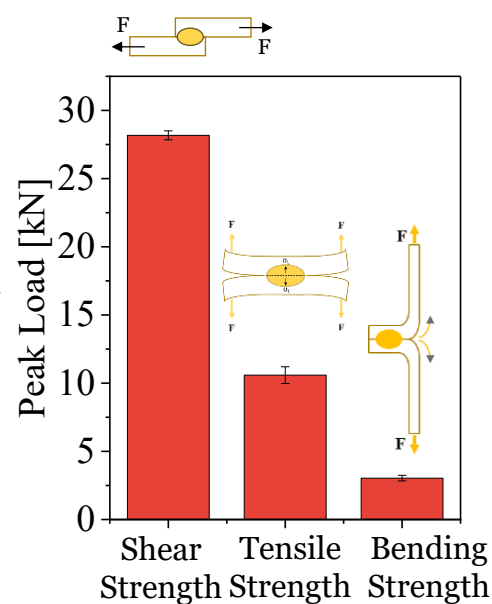
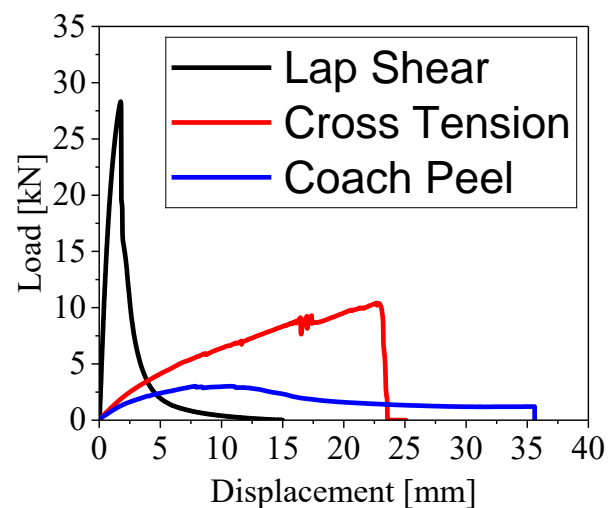
MWS: Minimum Weld Size ( $4\sqrt{t} \approx 4.8$  mm)

# Seeger's Nominal Failure Surfaces:

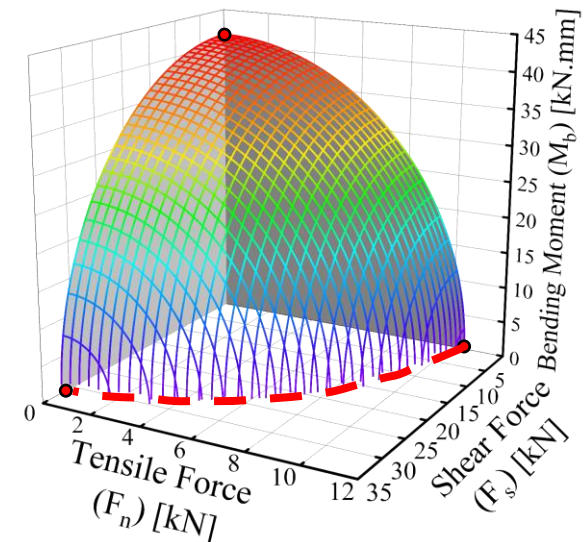
3G-980 FDWS



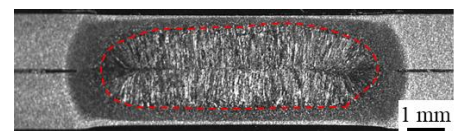
(nugget diameter:  $7.0 \pm 0.1$  mm)



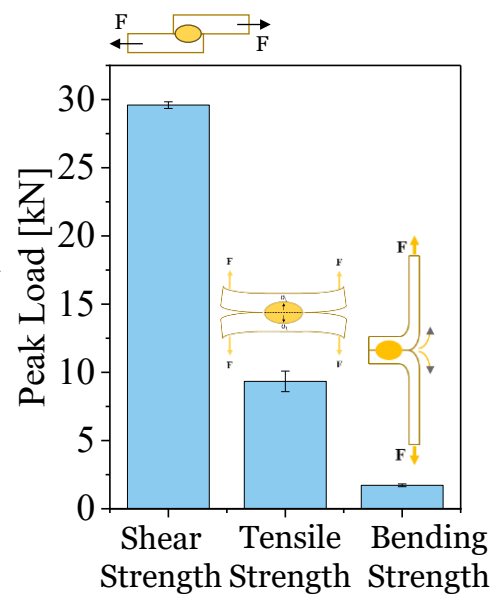
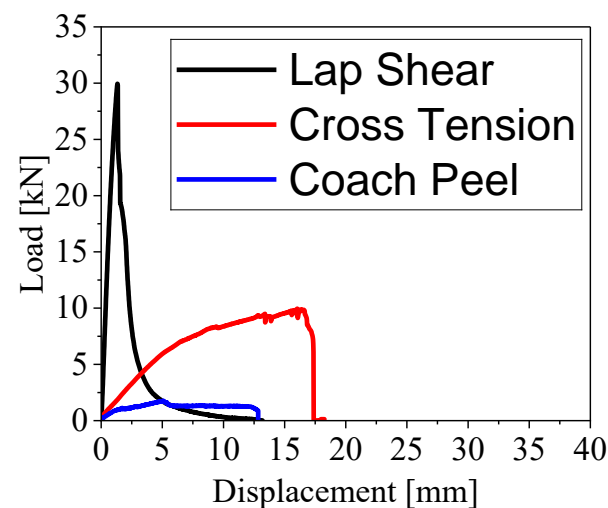
$$f_{Seeger} = \left\{ \left( \frac{f_s}{29.429} \right)^2 + \left( \frac{f_n}{10.324} \right)^2 + \left( \frac{b_m}{41.407} \right)^2 \right\} = 1$$



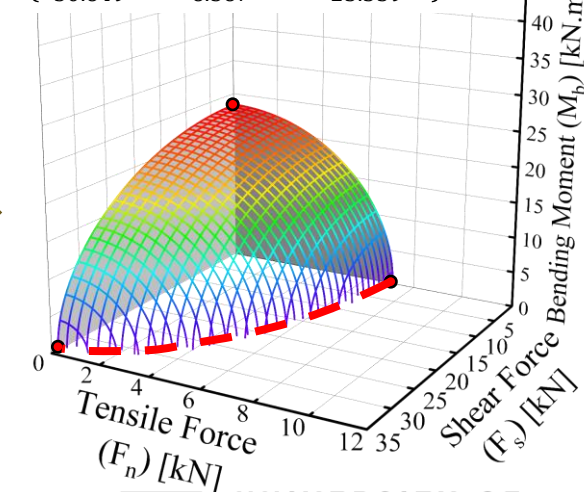
3G-1180 FDWS



(nugget diameter:  $7.0 \pm 0.2$  mm)

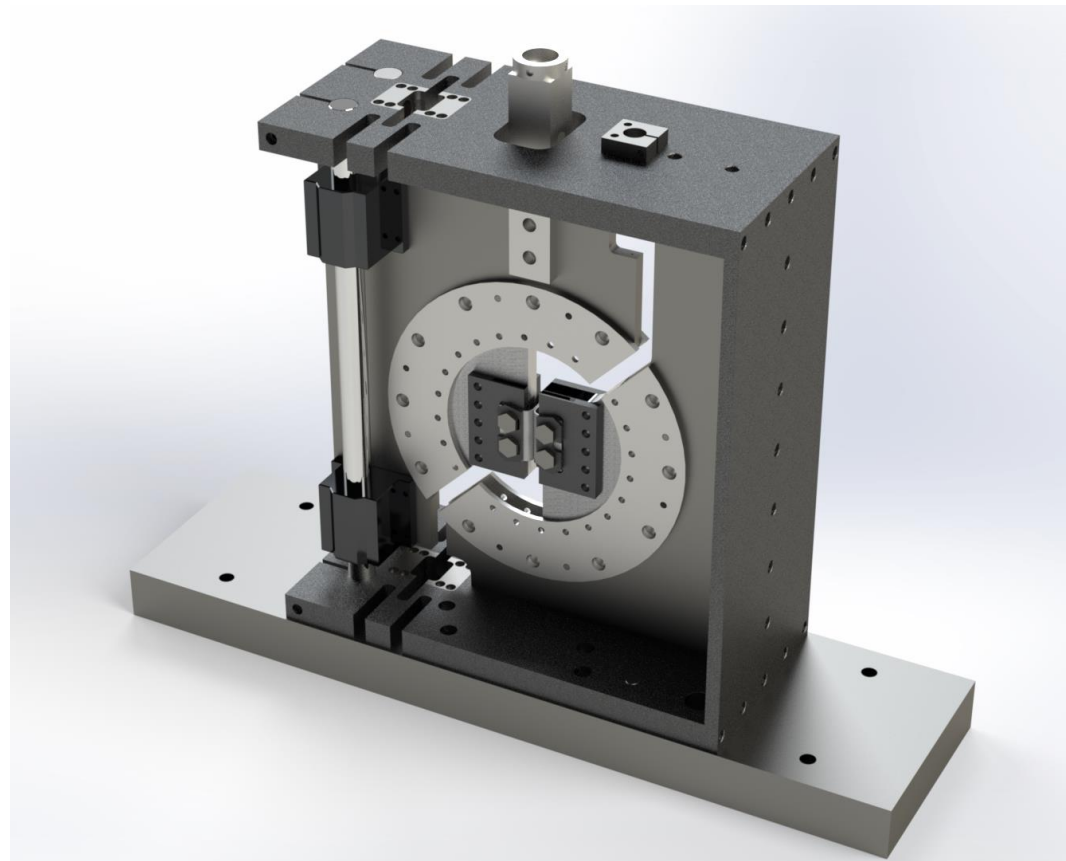
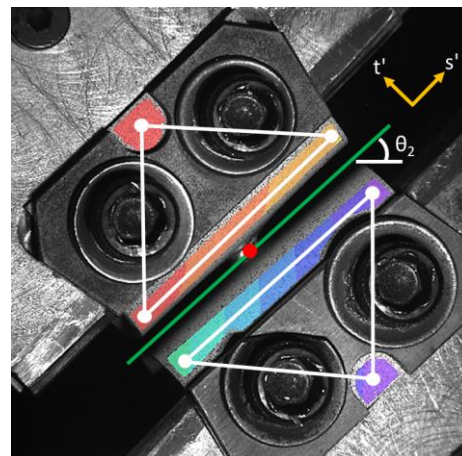
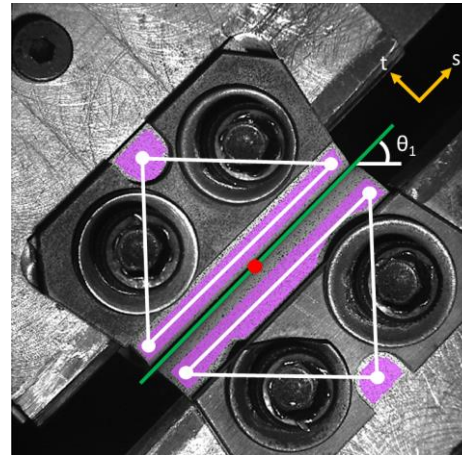
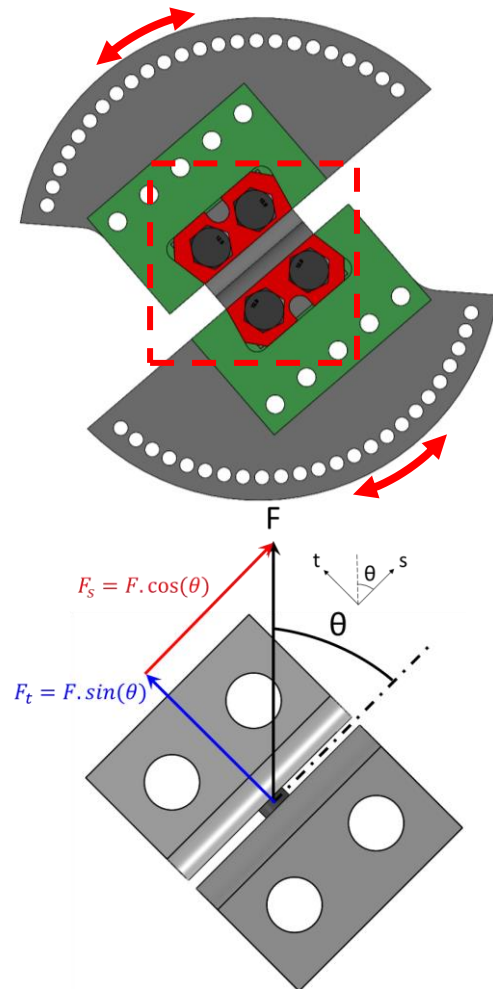


$$f_{Seeger} = \left\{ \left( \frac{f_s}{30.649} \right)^2 + \left( \frac{f_n}{6.567} \right)^2 + \left( \frac{b_m}{23.359} \right)^2 \right\} = 1$$

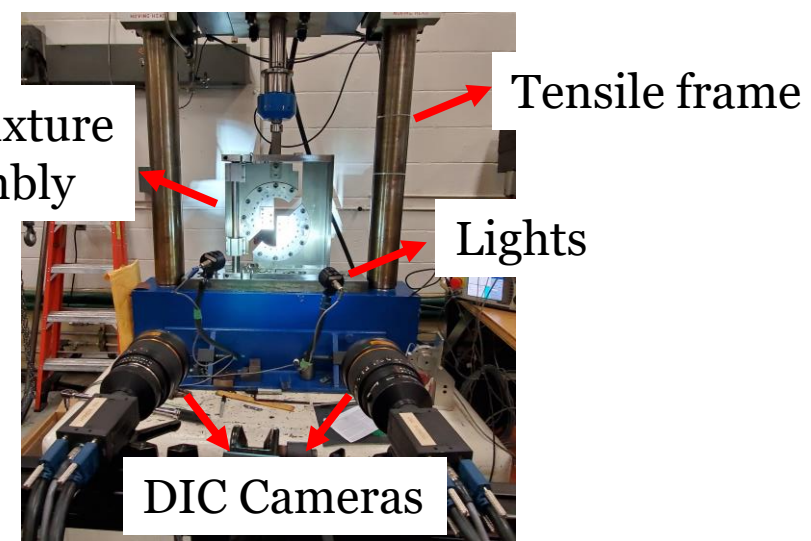
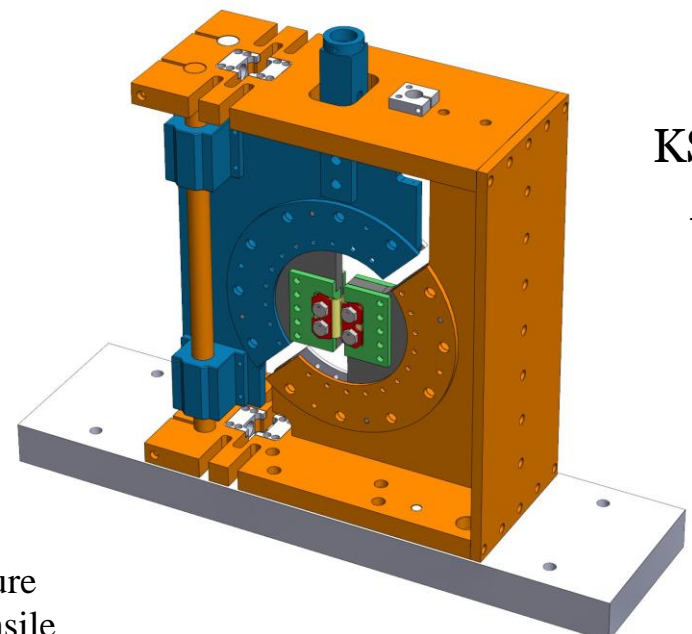
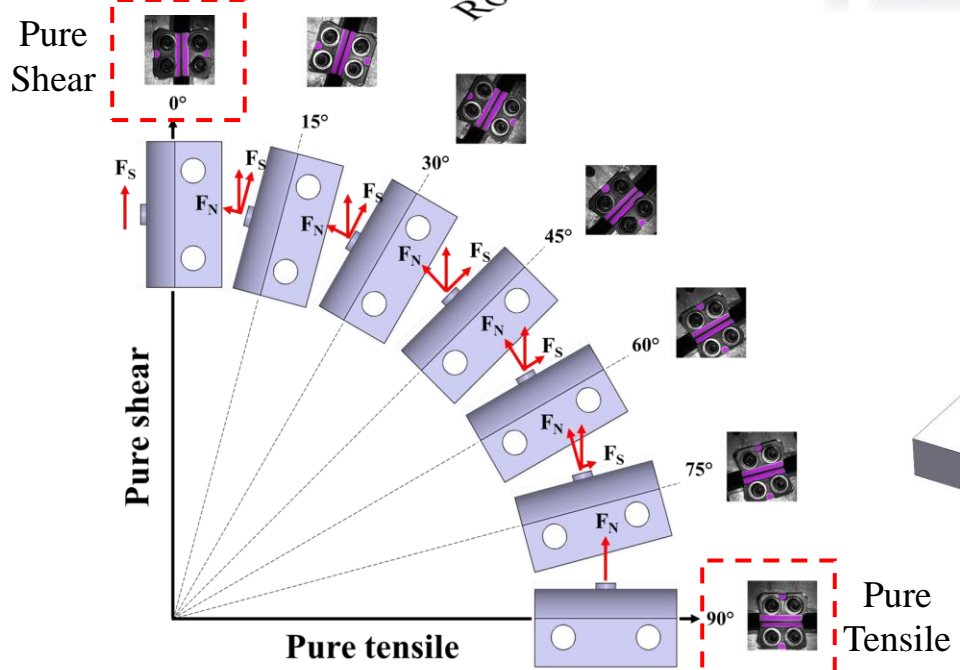
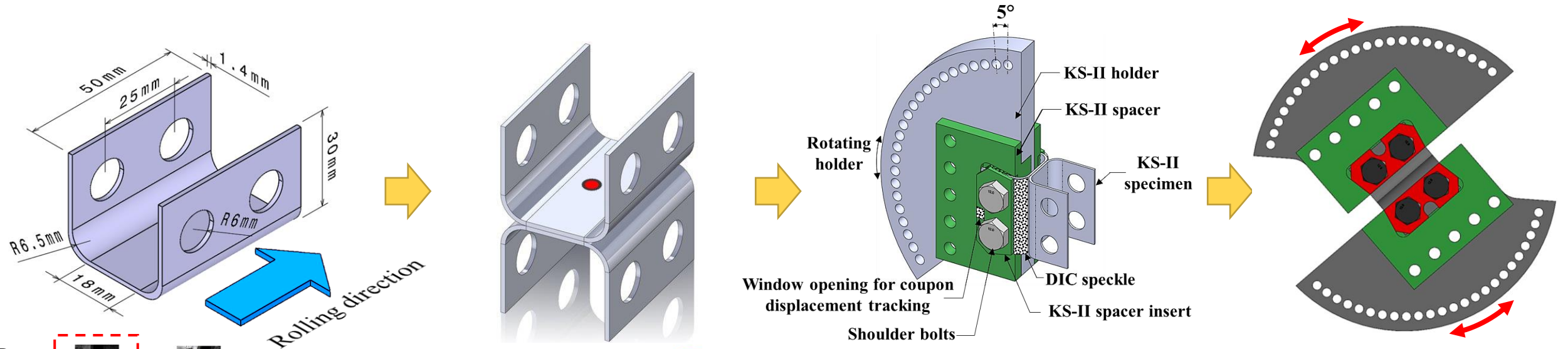


## Task 2

# Mechanical Performance of Spot Welds Under Shear-tensile Loading Condition: The KS-II Test

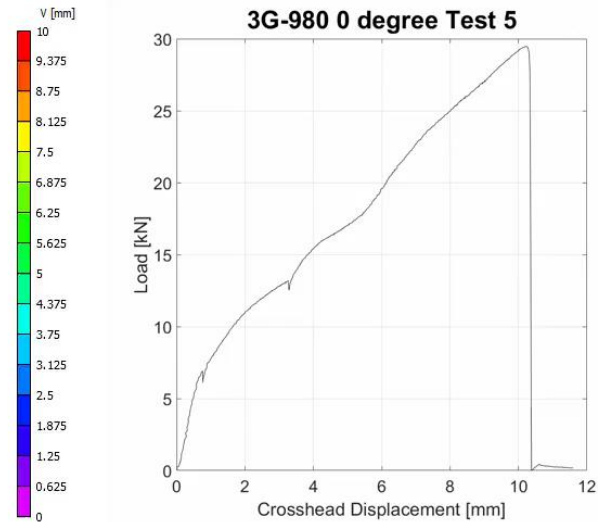
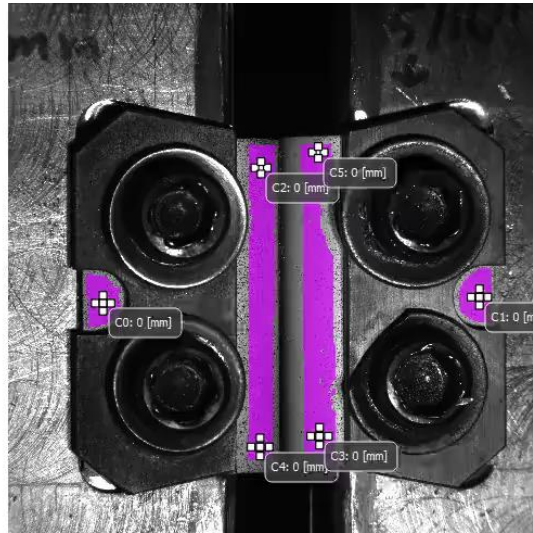


# Combined Shear-tensile Loading Methodology: The KS-II Test

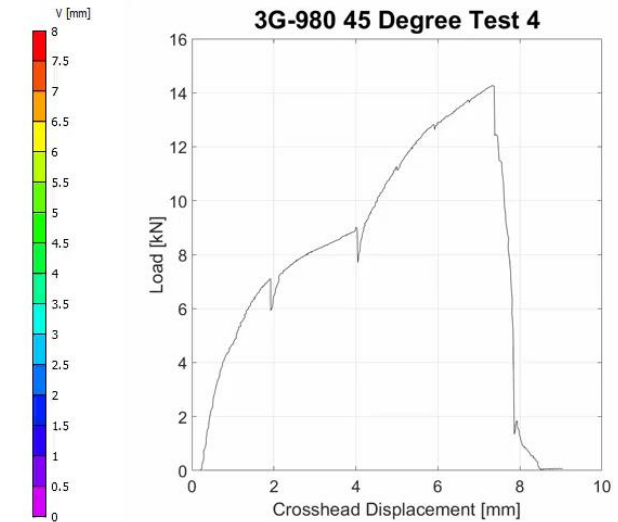
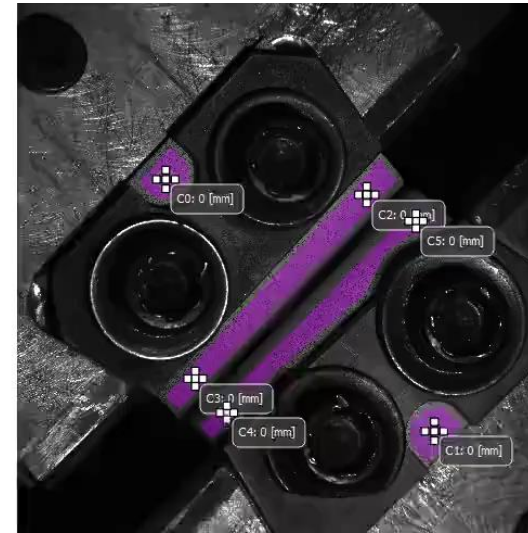


# Coupon Rotation and Slippage During KS-II Tests:

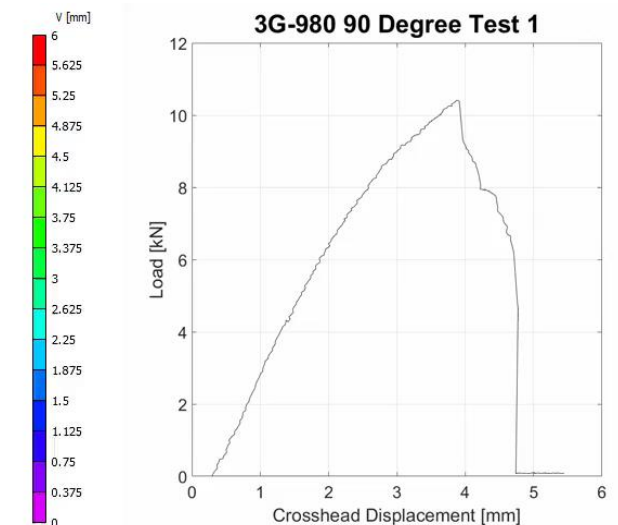
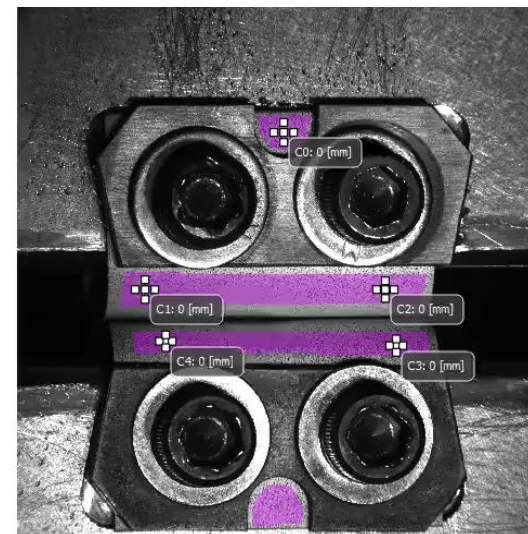
## 3G-980 KS-II 0° (Pure shear)



## 3G-980 KS-II 45°



## 3G-980 KS-II 90° (Pure tensile)

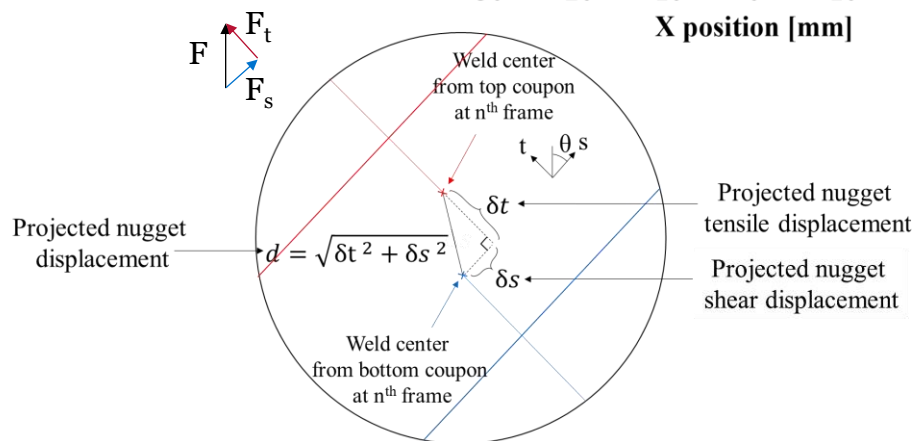
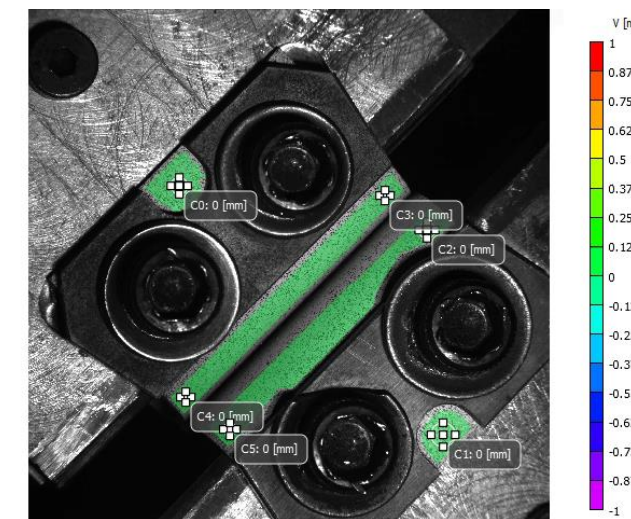
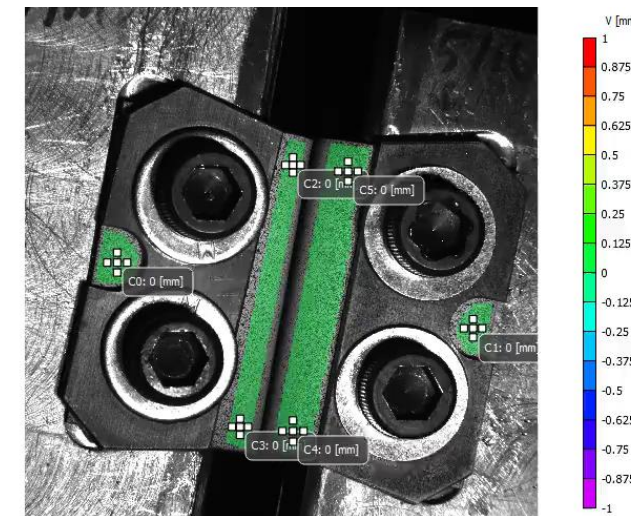
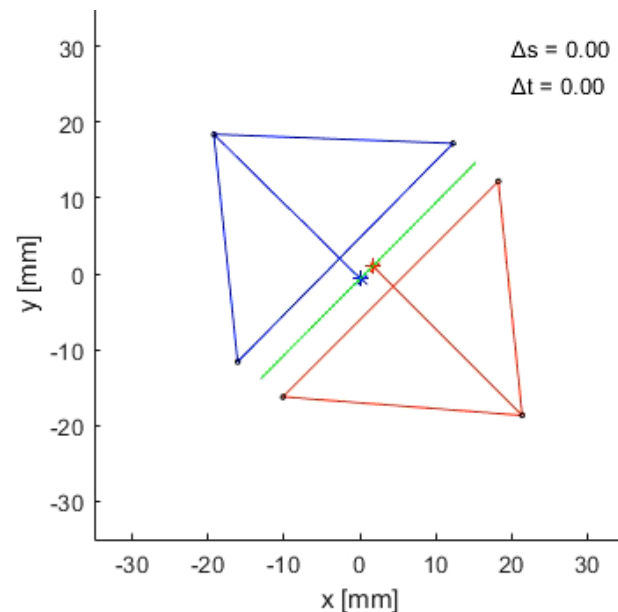
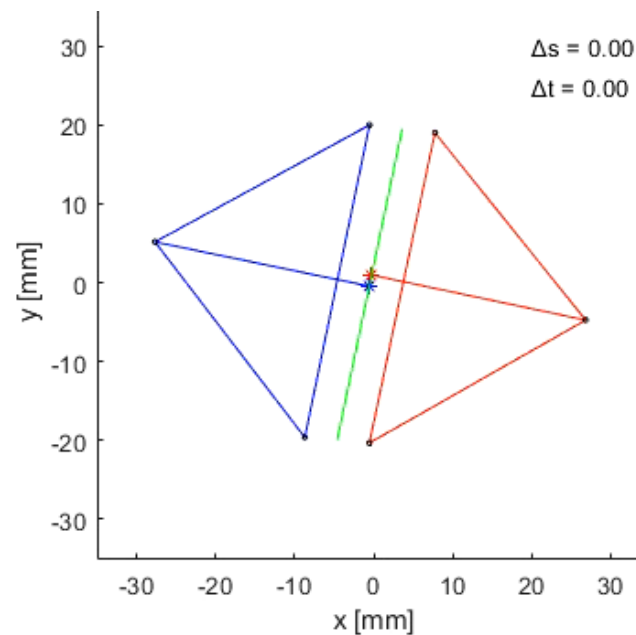
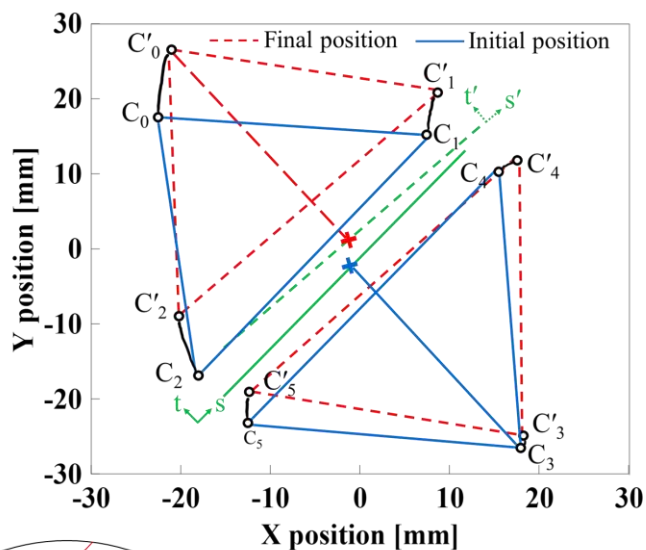
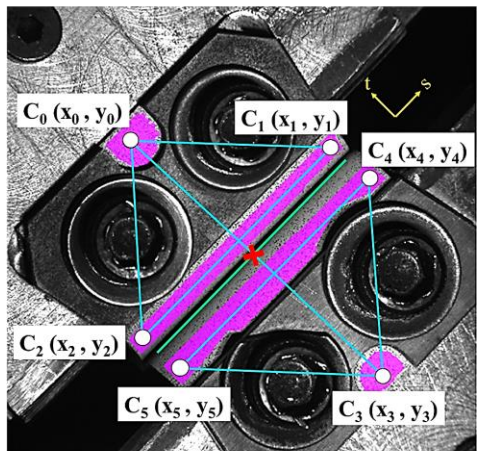


Slippage and rotation of the KS-II coupons were **noticeable** in **shear-dominated** loading orientations

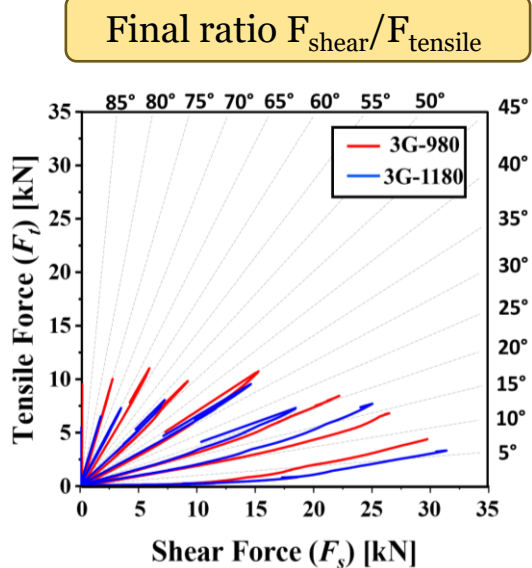
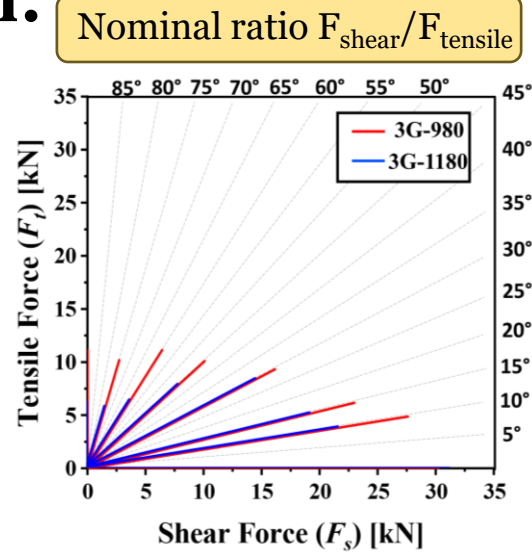
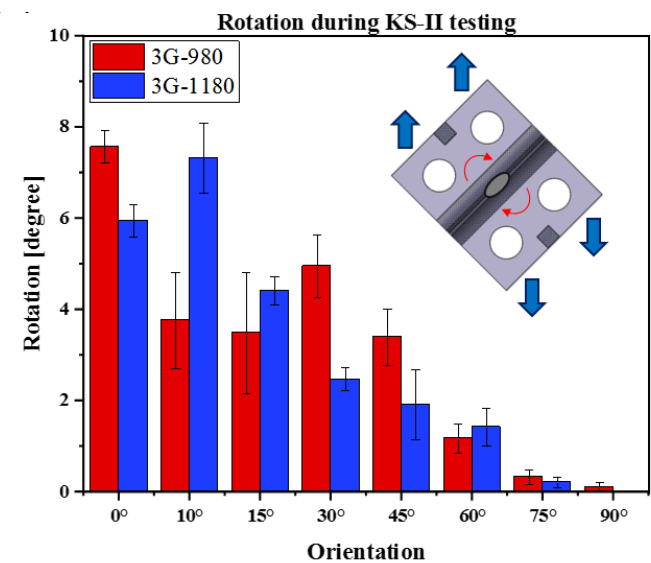
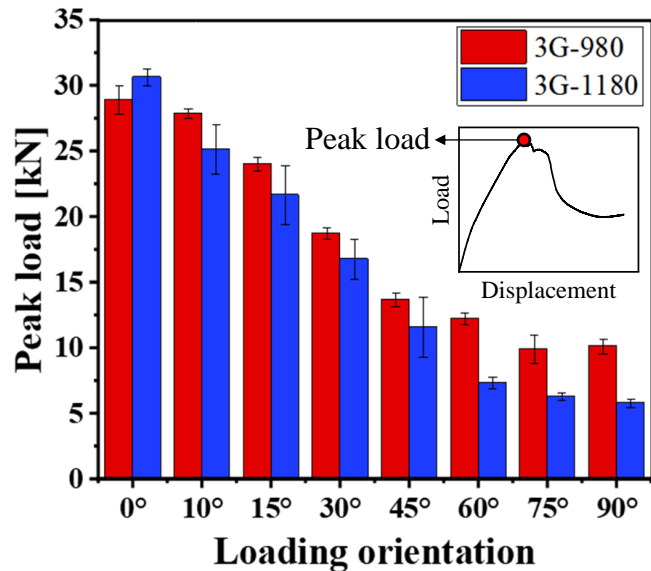
Slippage and rotation of the KS-II coupons were **negligible** in **tensile-dominated** loading orientations

# Developed Triangulation Method:

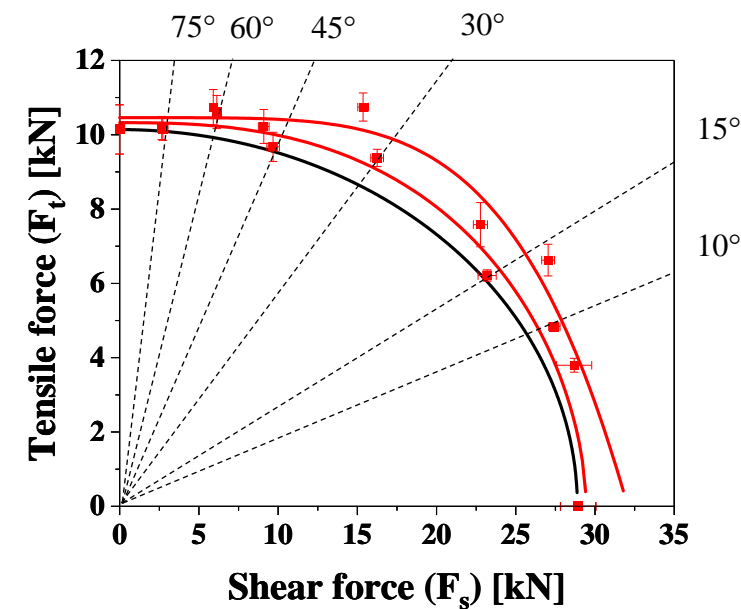
Instantaneous tracking of nugget orientation



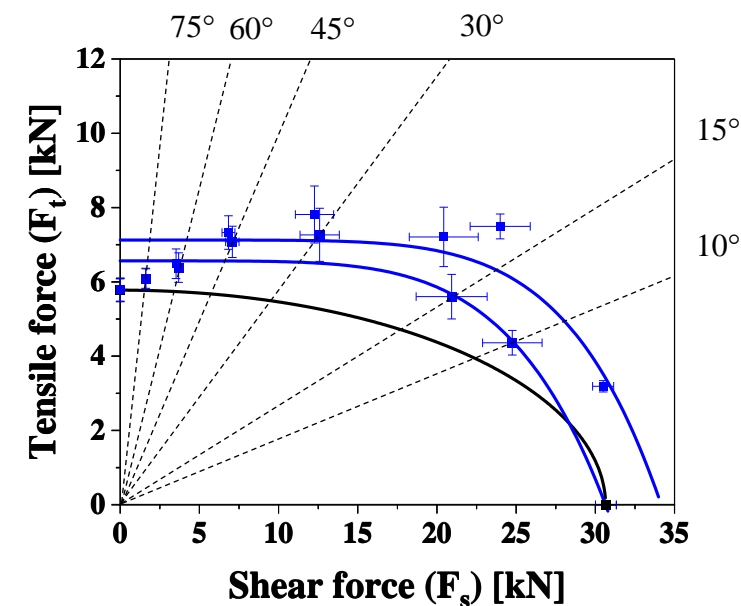
# Shear-tensile Failure Loci:



3G-980



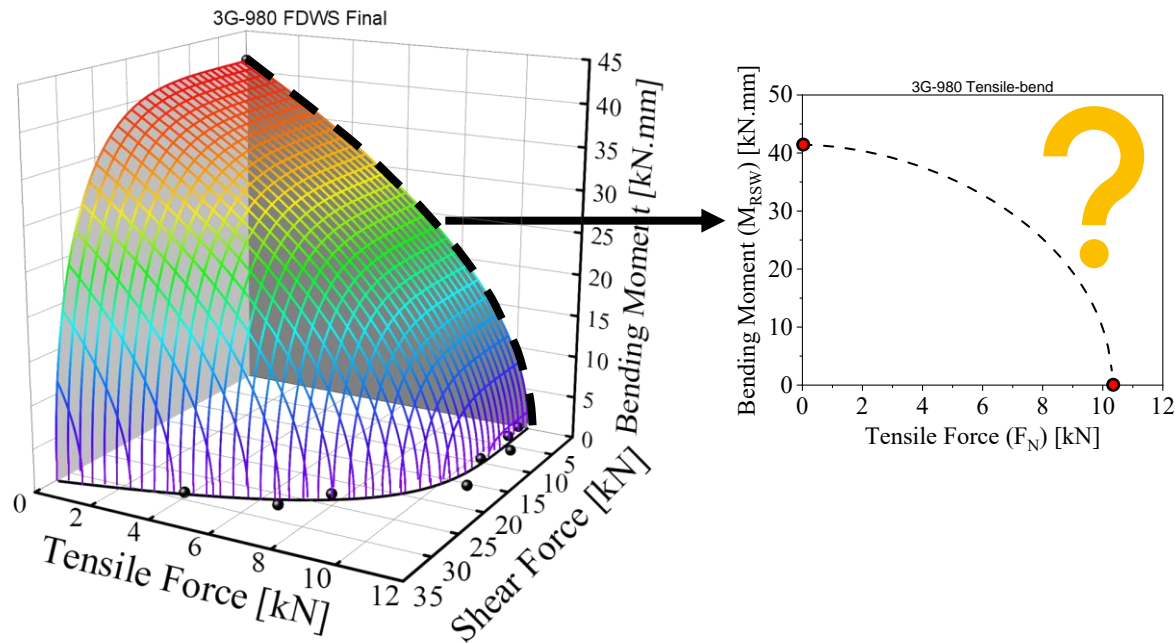
3G-1180



# Updated Failure Surface:

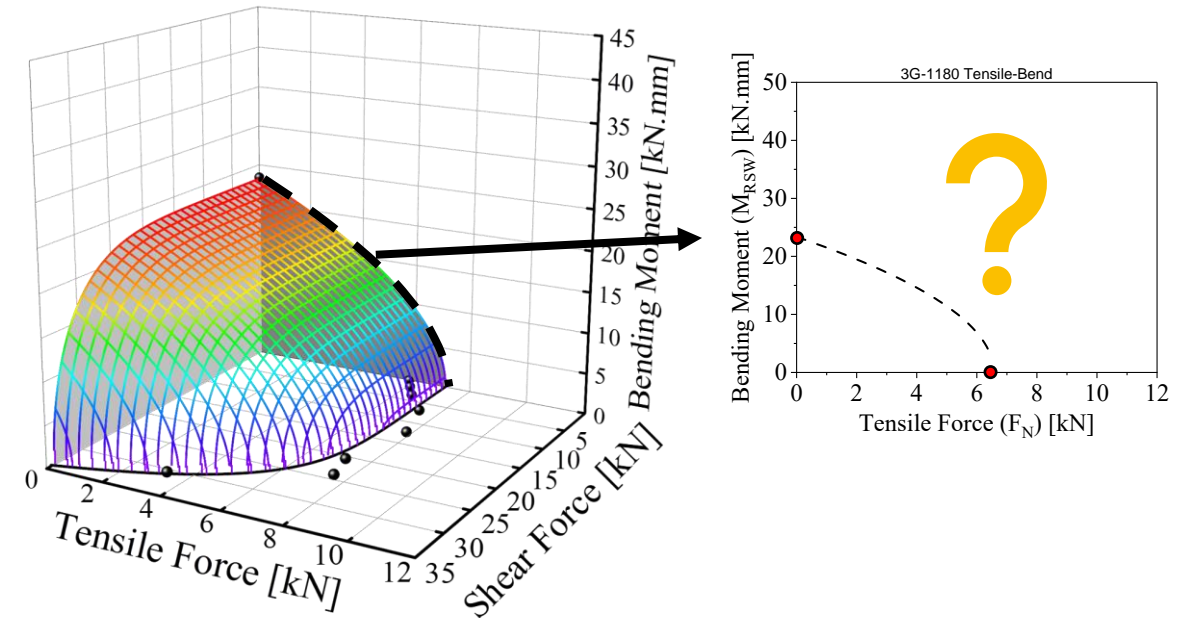
## 3G-980 FDWS

$$f_{Seeger} = \left\{ \left( \frac{f_s}{29.429} \right)^{2.616} + \left( \frac{f_n}{10.324} \right)^{1.819} + \left( \frac{b_m}{41.407} \right)^2 \right\} = 1$$



## 3G-1180 FDWS

$$f_{Seeger} = \left\{ \left( \frac{f_s}{30.649} \right)^{5.222} + \left( \frac{f_n}{6.567} \right)^{1.123} + \left( \frac{b_m}{23.359} \right)^2 \right\} = 1$$



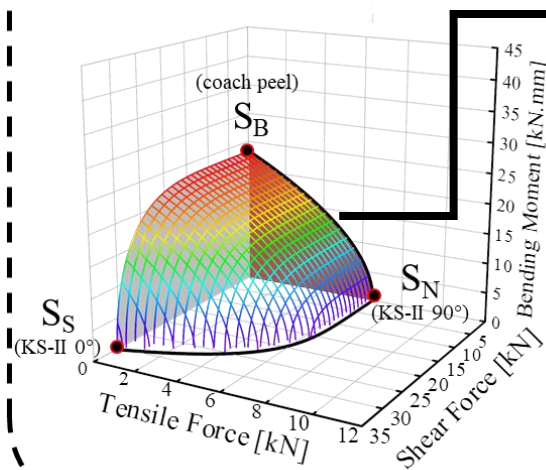
## Task 3

# Mechanical Performance of Spot Welds Under Tensile-bending Loading Condition: The Coach Peel Test

Conventional Three-dimensional Failure Surface Development for 3G-AHSS Resistance Spot Welds

$$f = \left[ \frac{F_s}{S_s} \right]^a + \left[ \frac{F_n}{S_n} \right]^b + \left[ \frac{M_b}{S_b} \right]^c \leq 1$$

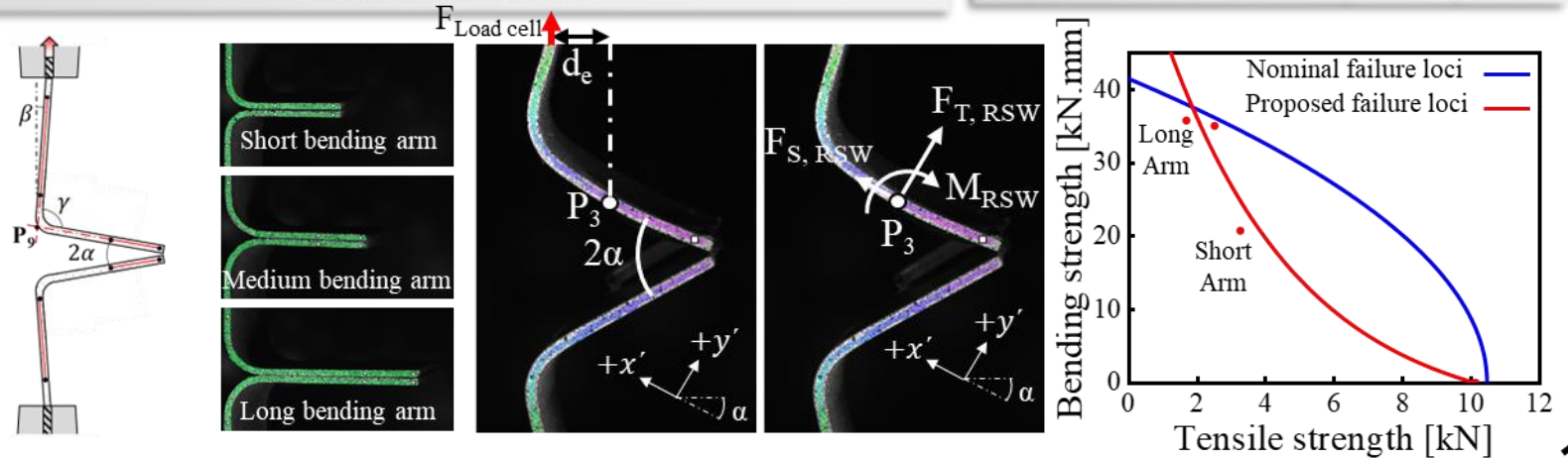
Shear Strength      Tensile Strength      Bending Strength



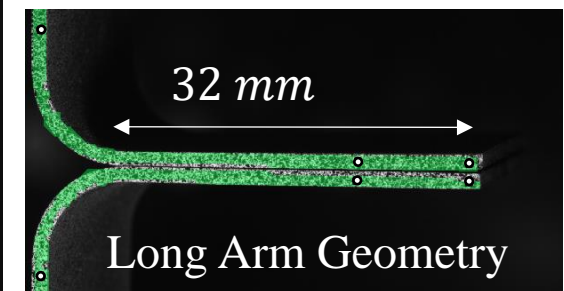
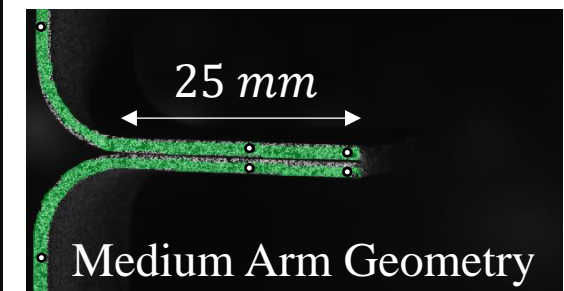
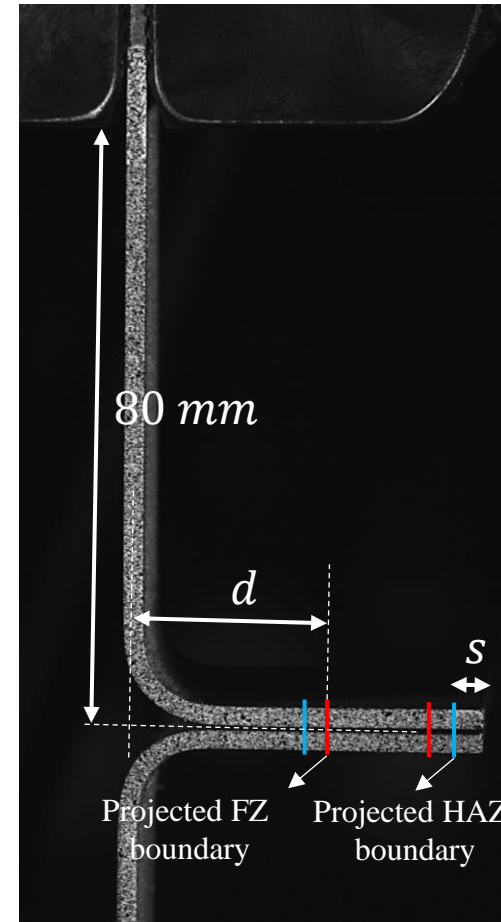
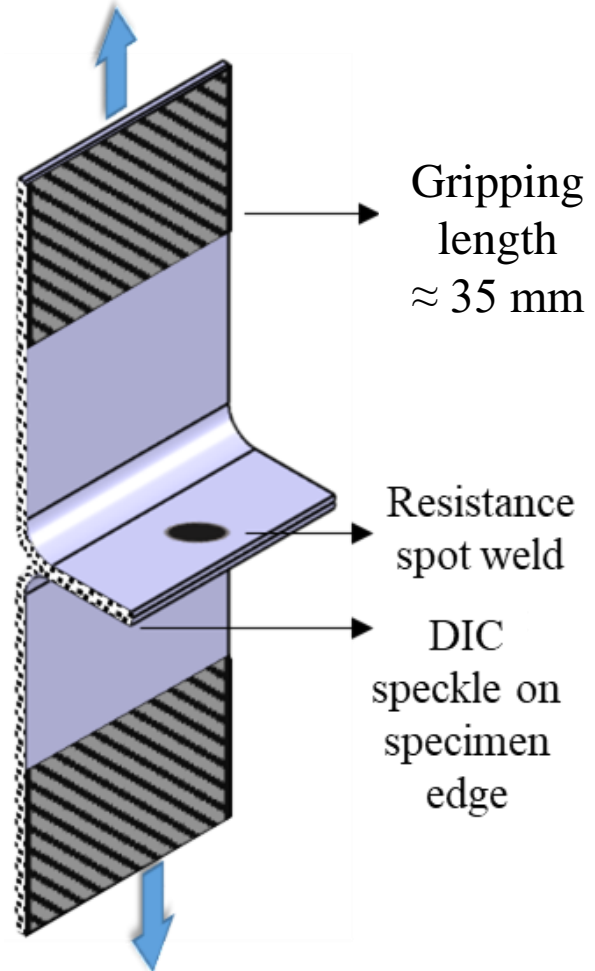
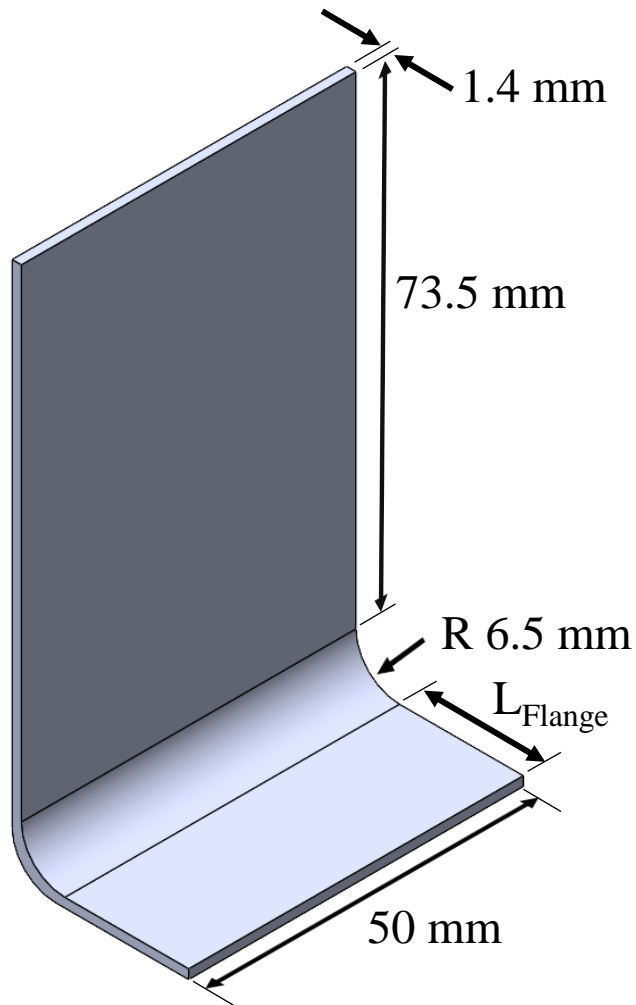
Evaluating Failure Surface Predictions in Bending Dominated Loading Conditions

Tracking of Tensile Force and Bending Arm Evolution Using Different Coach Peel Geometries

Improved Failure Locus Functional Form Proposed in Tensile-Bending Loading Plane

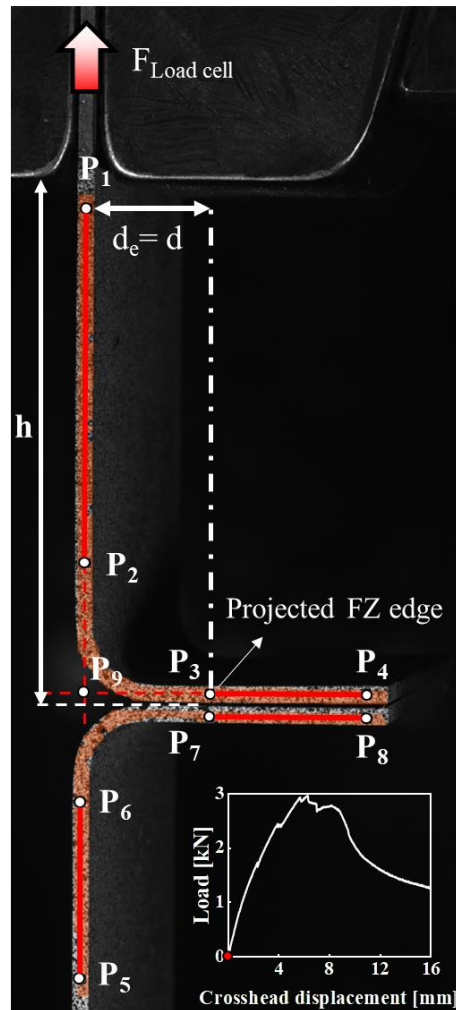


# Combined Tensile-bending Loading Methodology: The Coach Peel Test

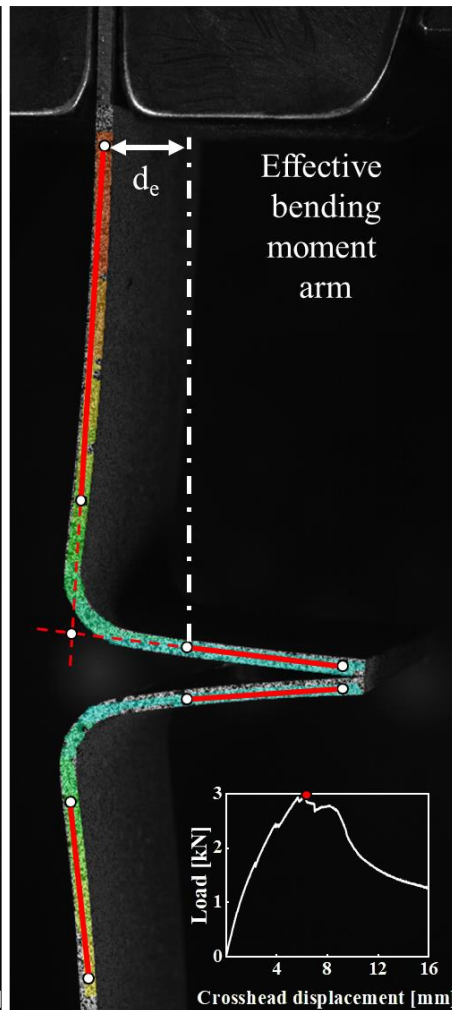


## An example of coupon deformation during coach peel tests:

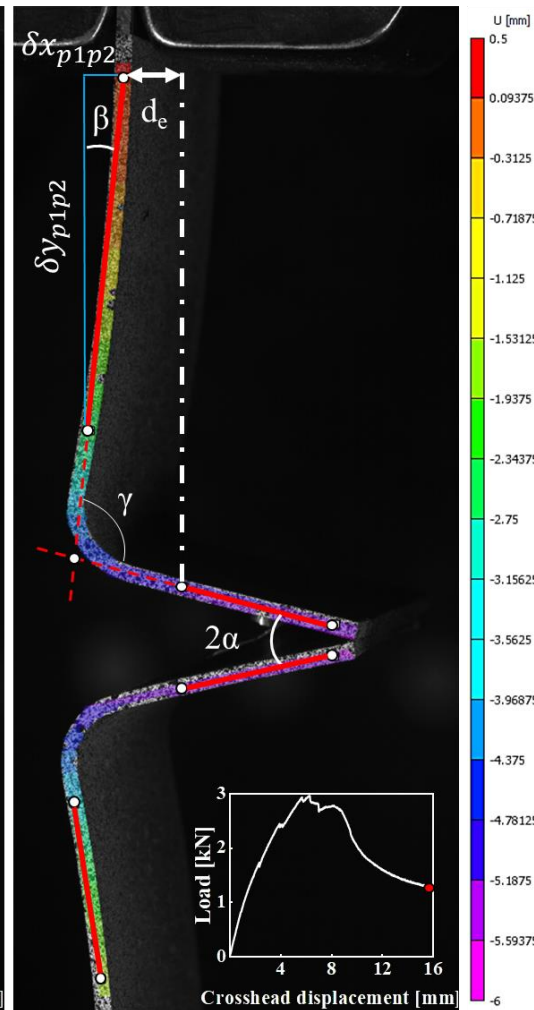
First frame



Peak load frame

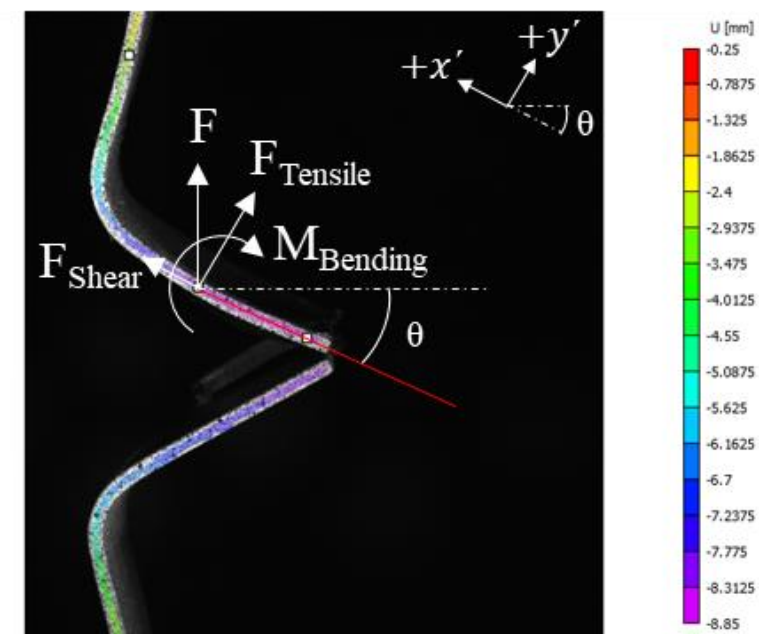
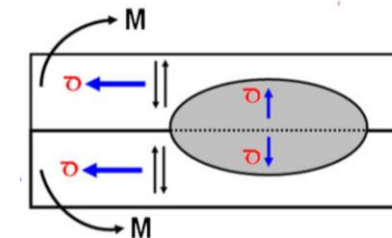


Final frame

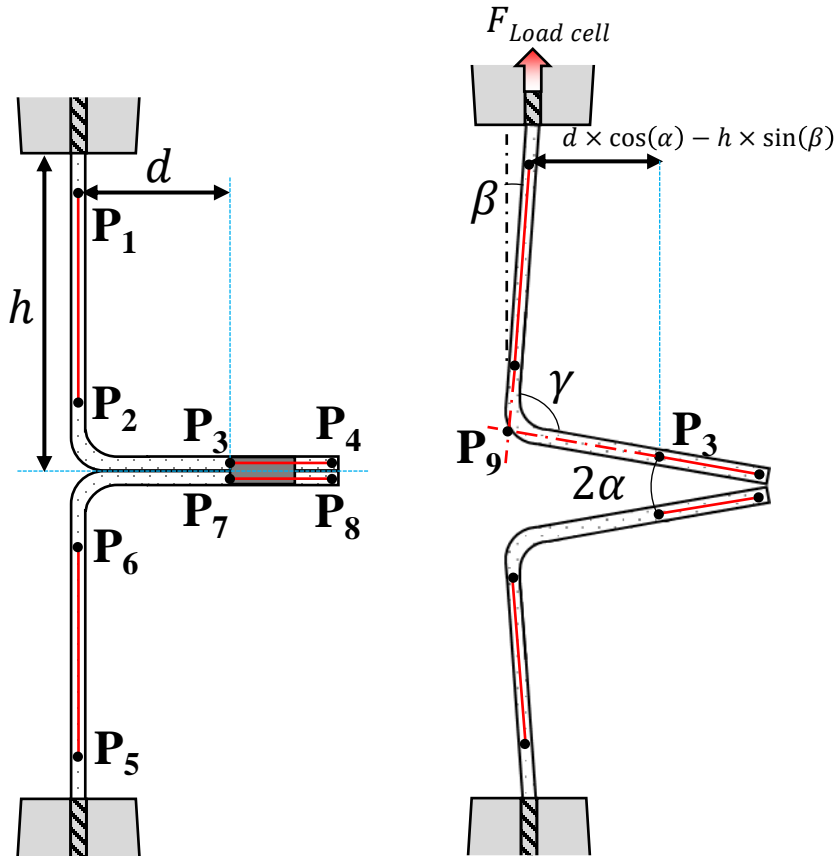


The evolution of the following loads/moments should be considered:

- Bending Moment
- Tensile Stress
- Transverse Shear Stress



### Static force equilibrium analysis on coach peel coupons:



Shear force

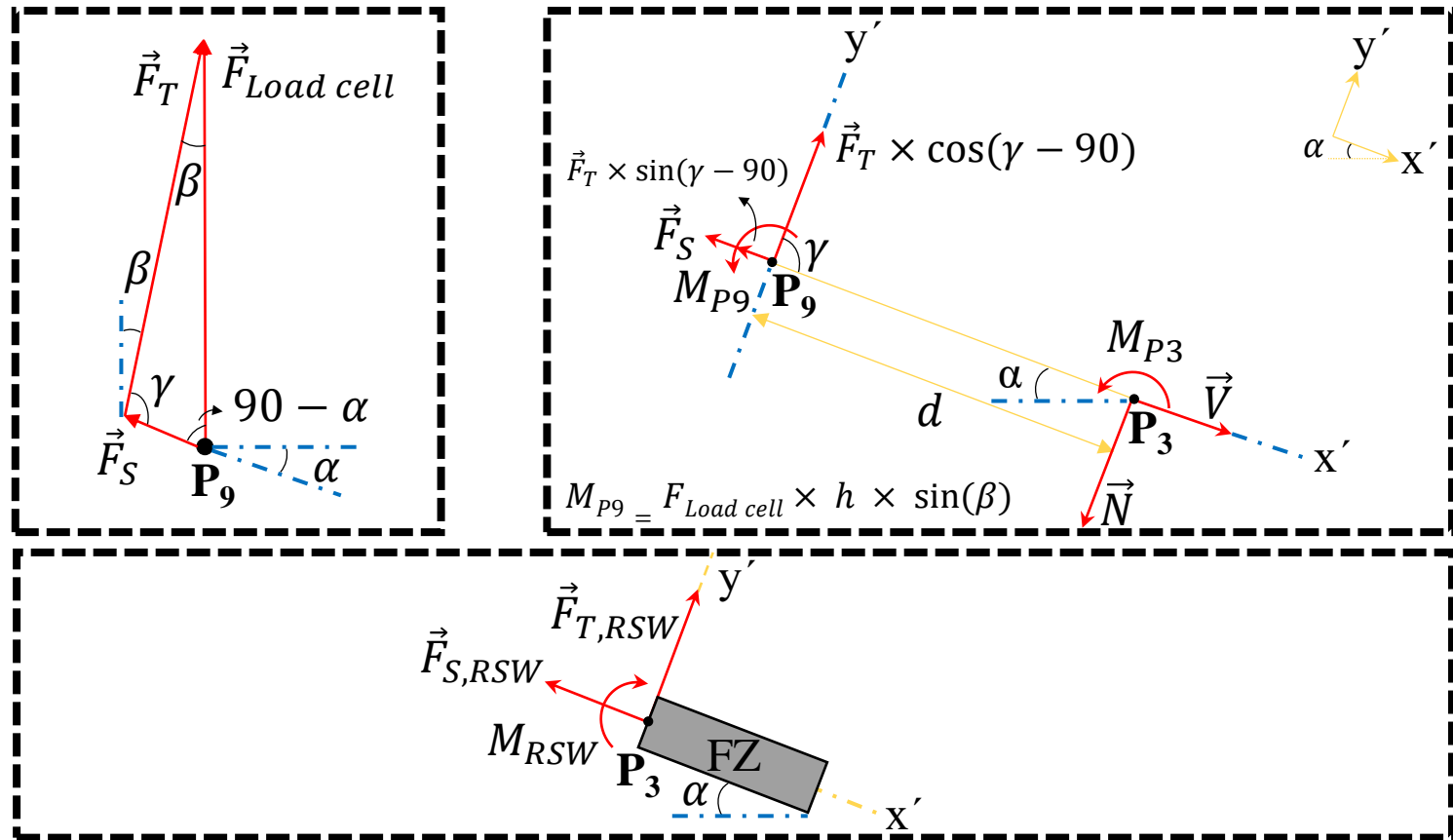
$$|\vec{F}_{S,RSW}| = |\vec{V}| = \frac{|\vec{F}_{Load\ cell}|}{\sin(\gamma)} [\sin(\beta) - \cos(\alpha) \cos(\gamma)]$$

Tensile force

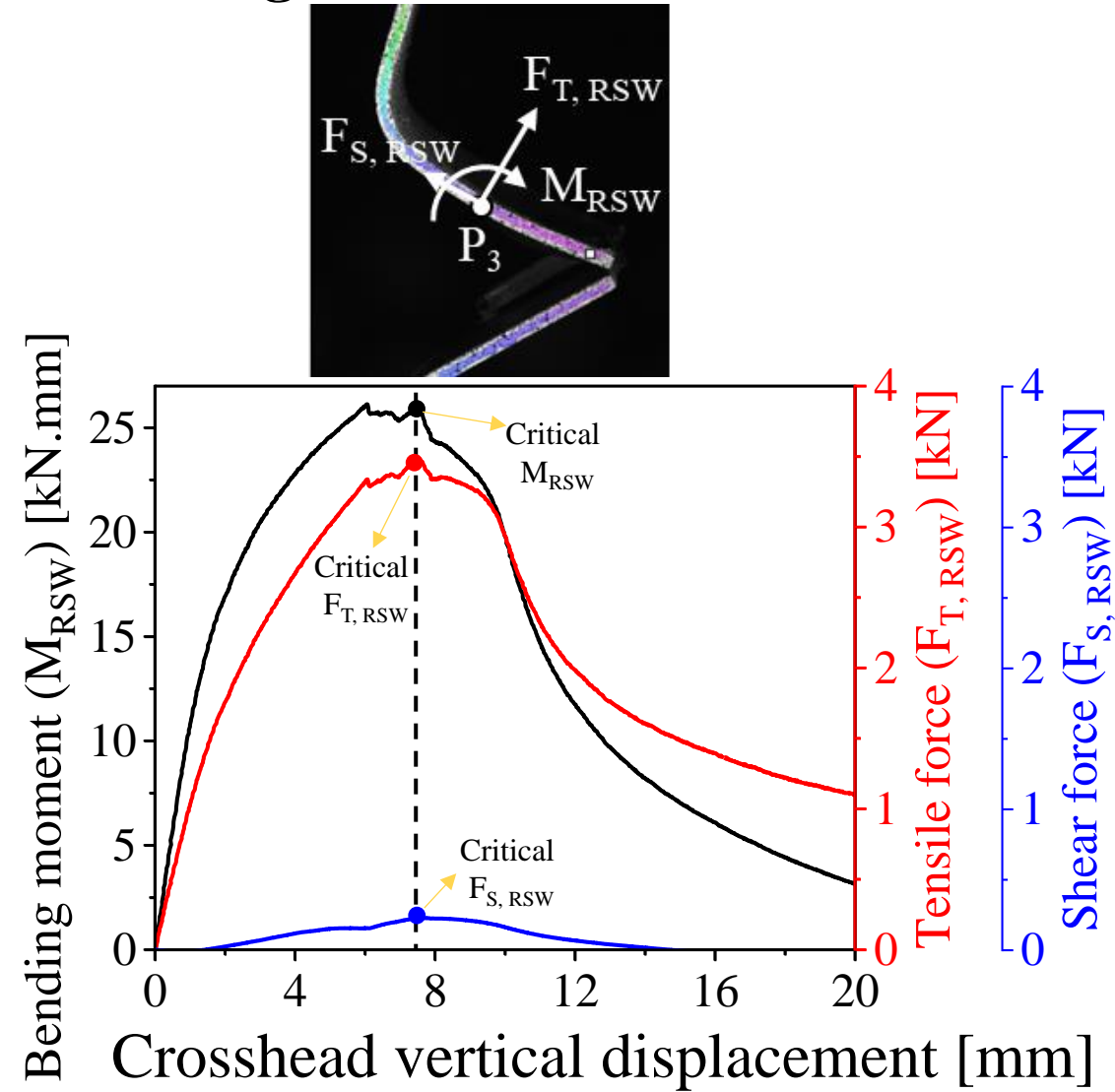
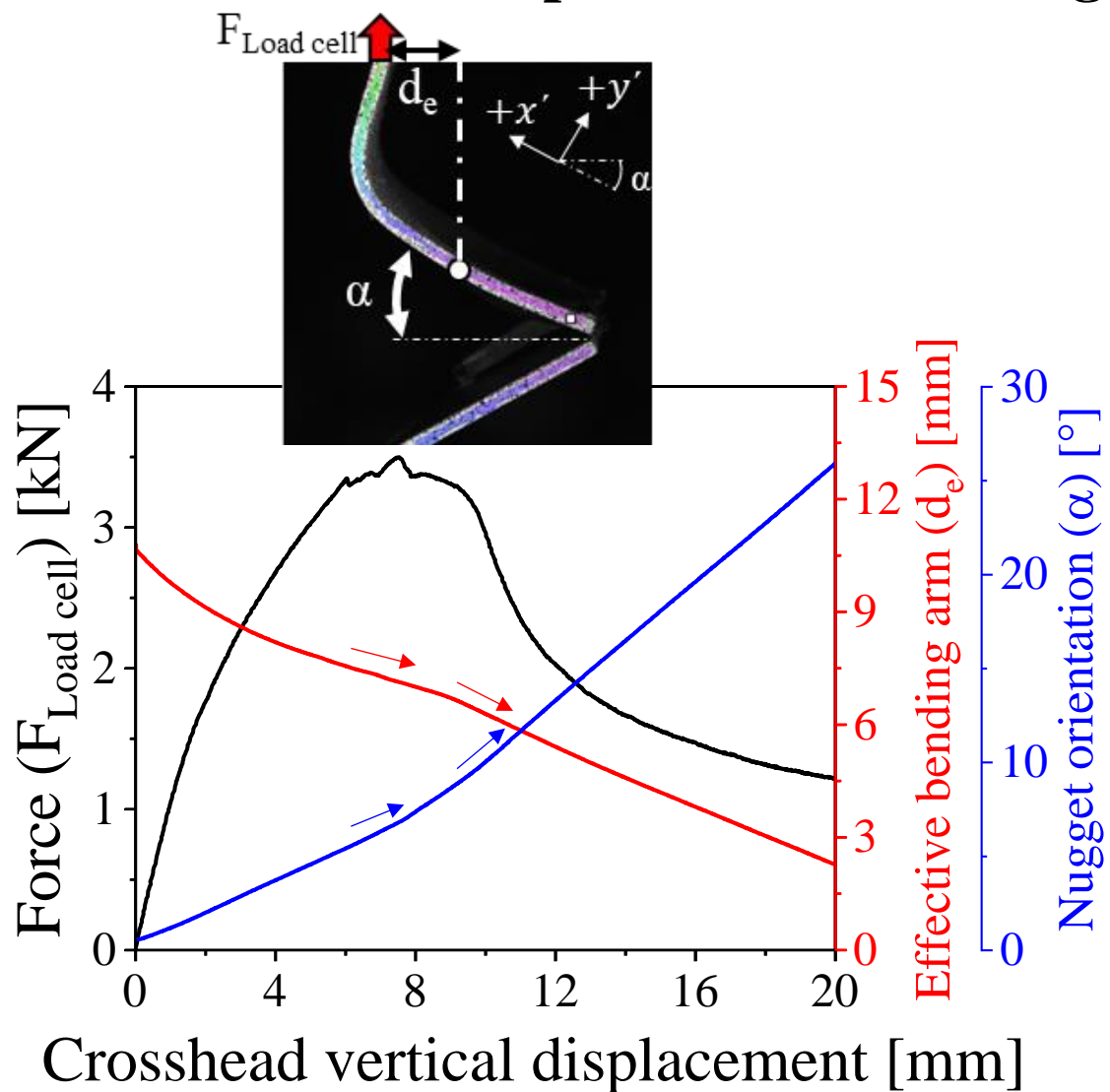
$$|\vec{F}_{T,RSW}| = |\vec{N}| = |\vec{F}_{Load\ cell}| \cos(\alpha)$$

Bending moment

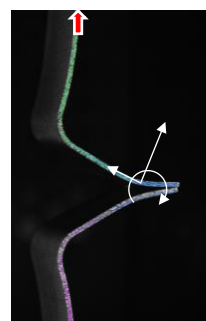
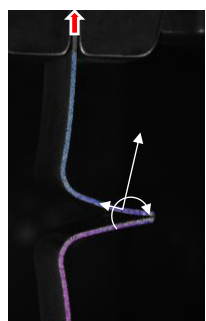
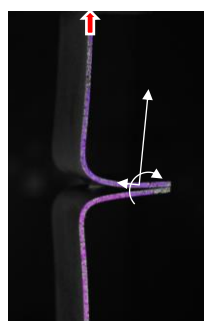
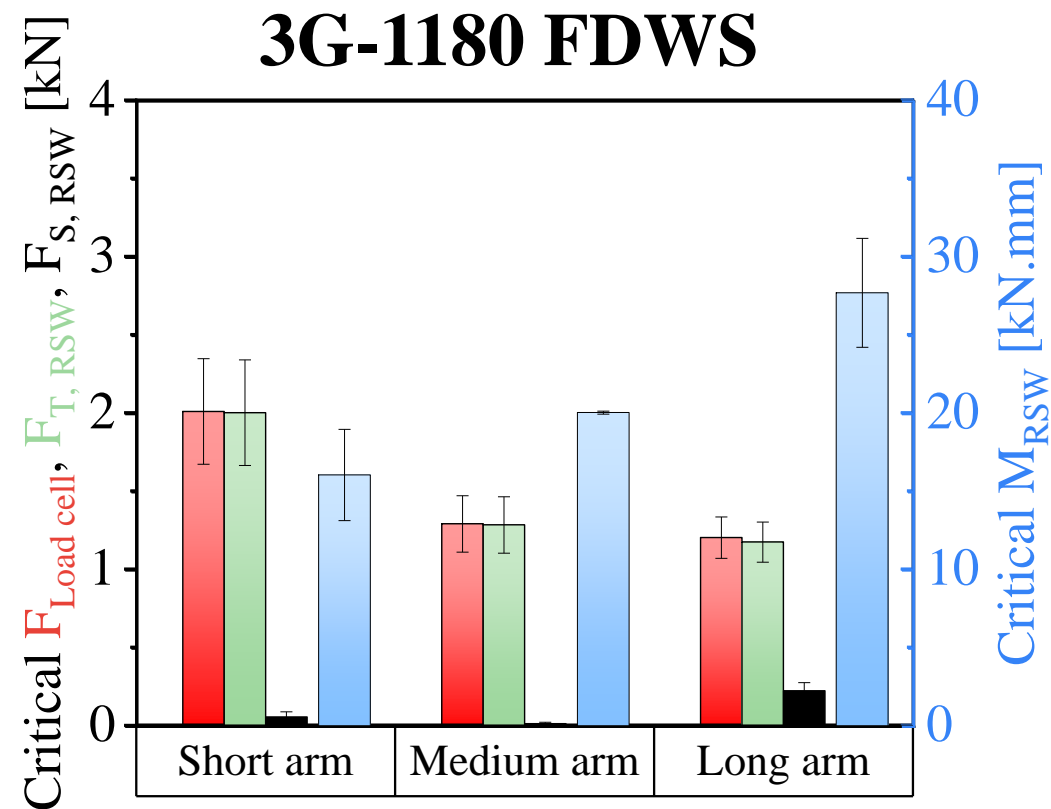
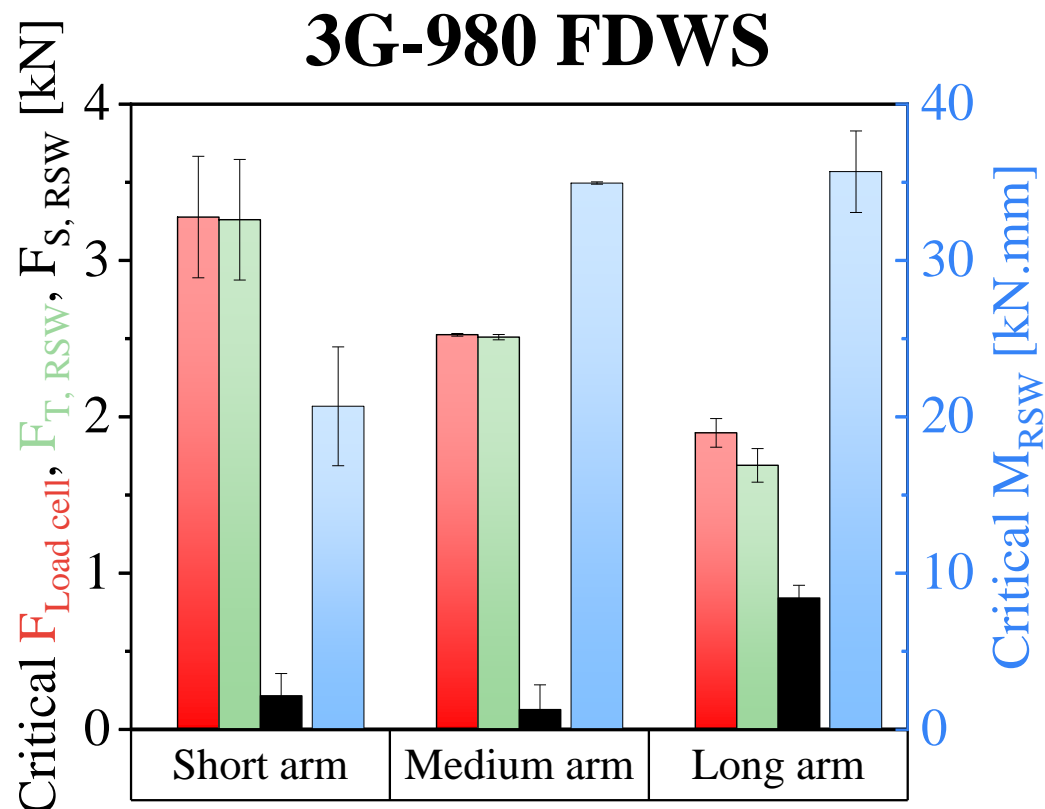
$$M_{RSW} = |\vec{F}_{Load\ cell}| \times [d \times \cos(\alpha) - h \times \sin(\beta)]$$



# Evolution of force components and bending moment during a CP test:

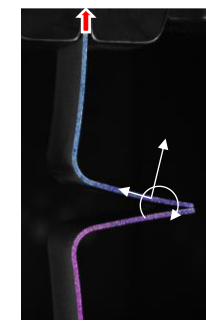
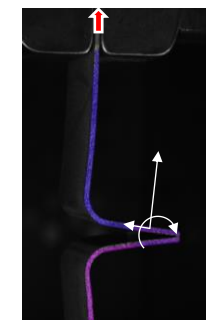
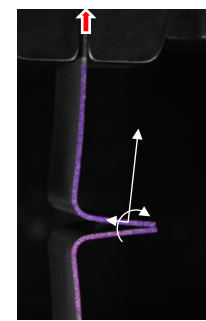


# Summary of critical loads/moments at the onset of failure during coach peel tests:


 $F_{\text{load cell}}$ 

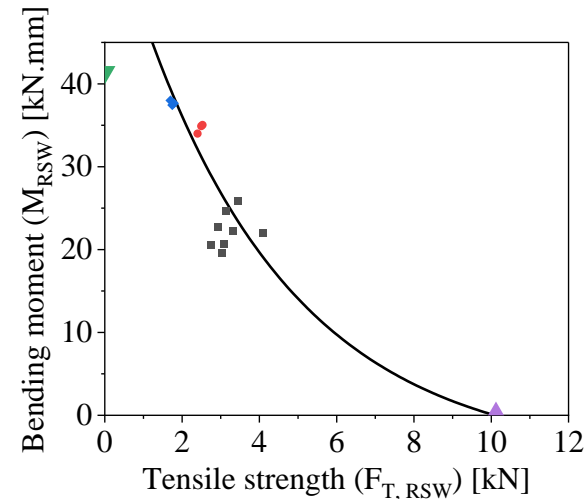
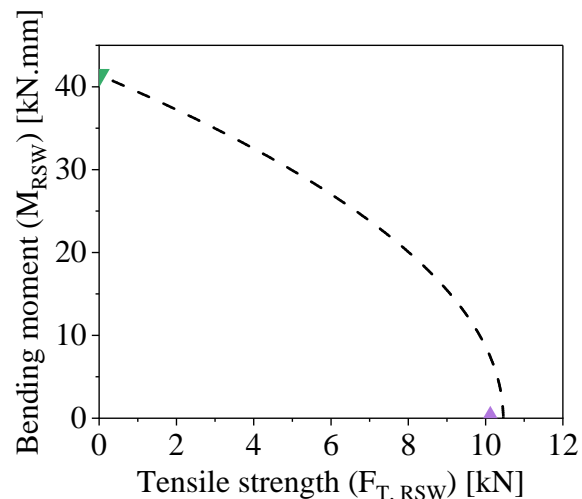
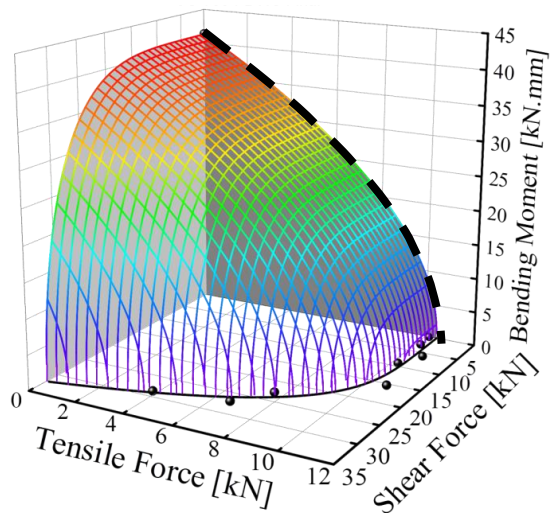
 $F_{T, RSW}$ 

 $M_{RSW}$ 

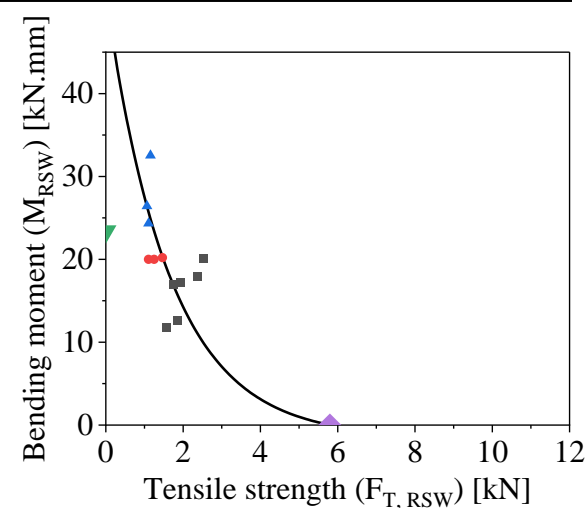
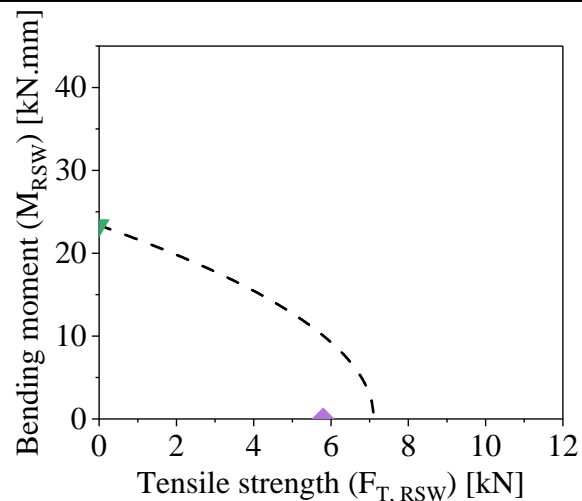
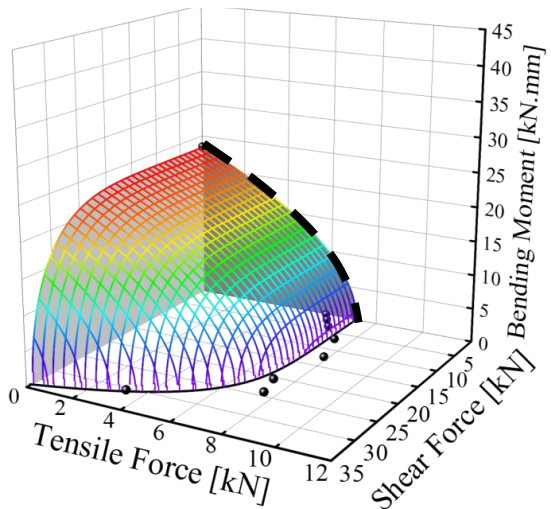
 $F_{S, RSW}$ 


# Assessing Seeger's Failure Loci in Tensile-bending Loading:

3G-980



3G-1180

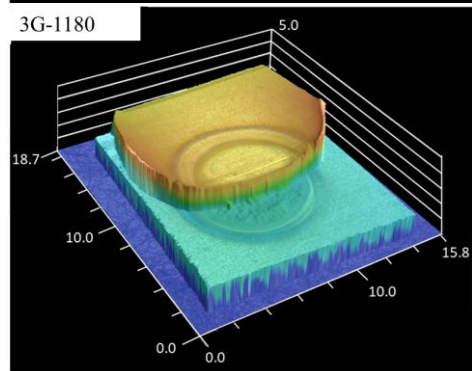
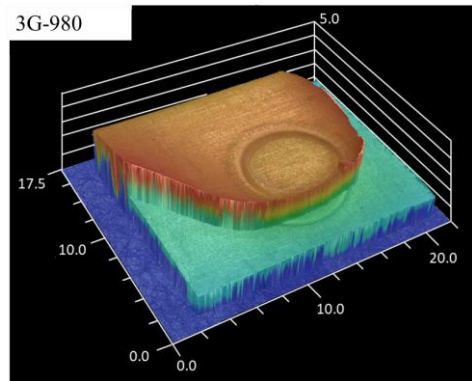


Results indicate an exponentially decreasing (concave) failure loci rather than the convex shape postulated by Seeger

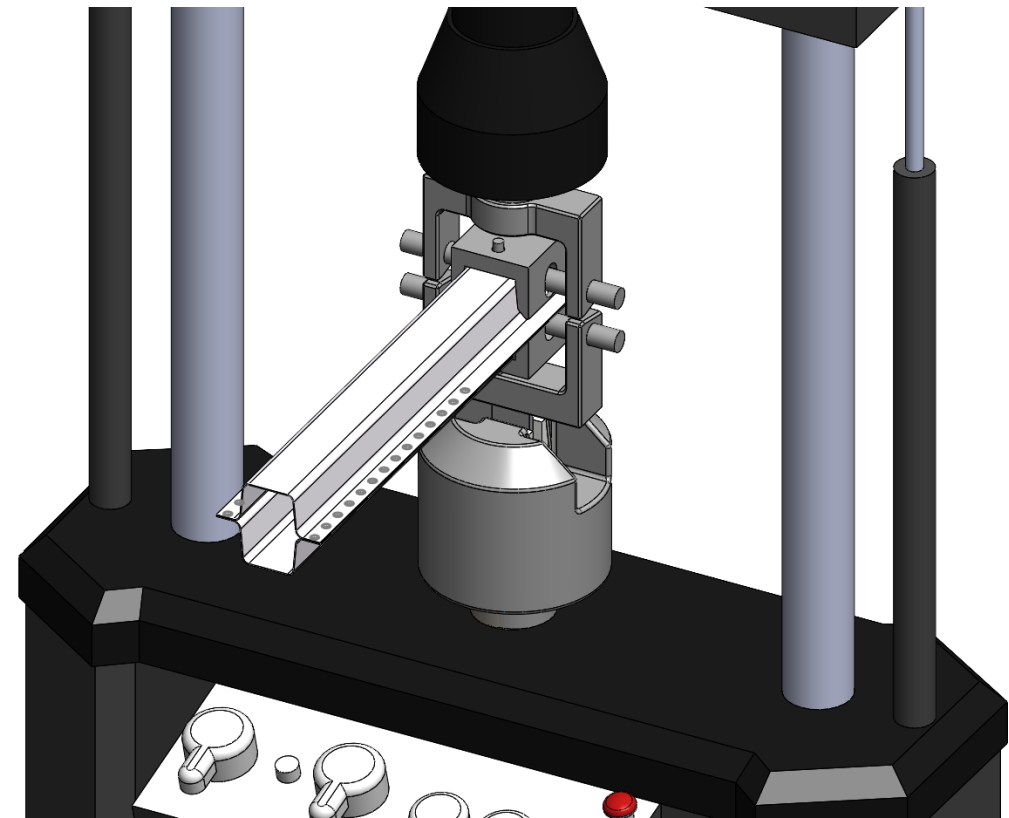
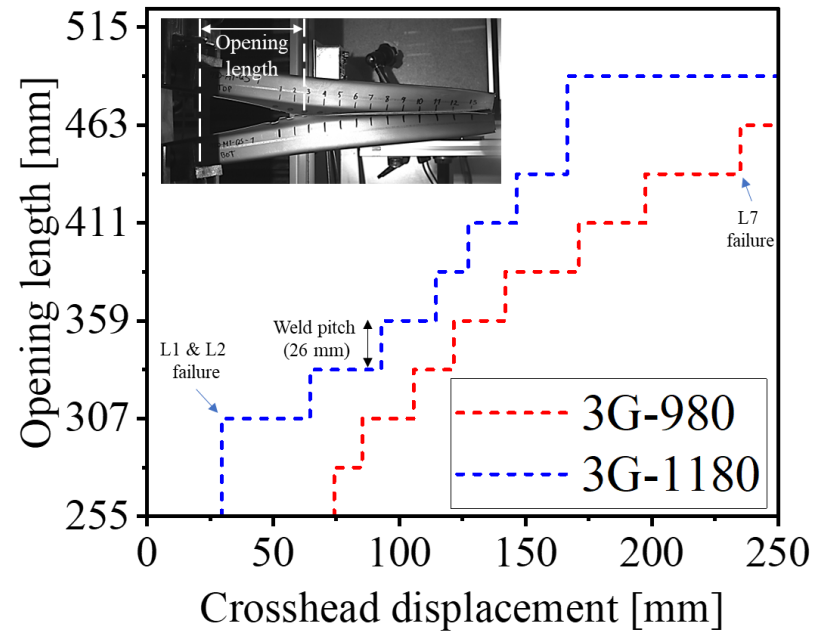
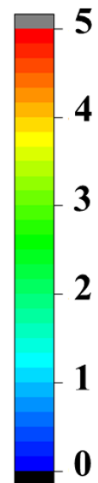
Coach peel (short arm)
  Coach peel (medium arm)
  Coach peel (long arm)
  Tensile strength (KS-II 90°)
  Bending strength ( $S_B$ )

## Task 4

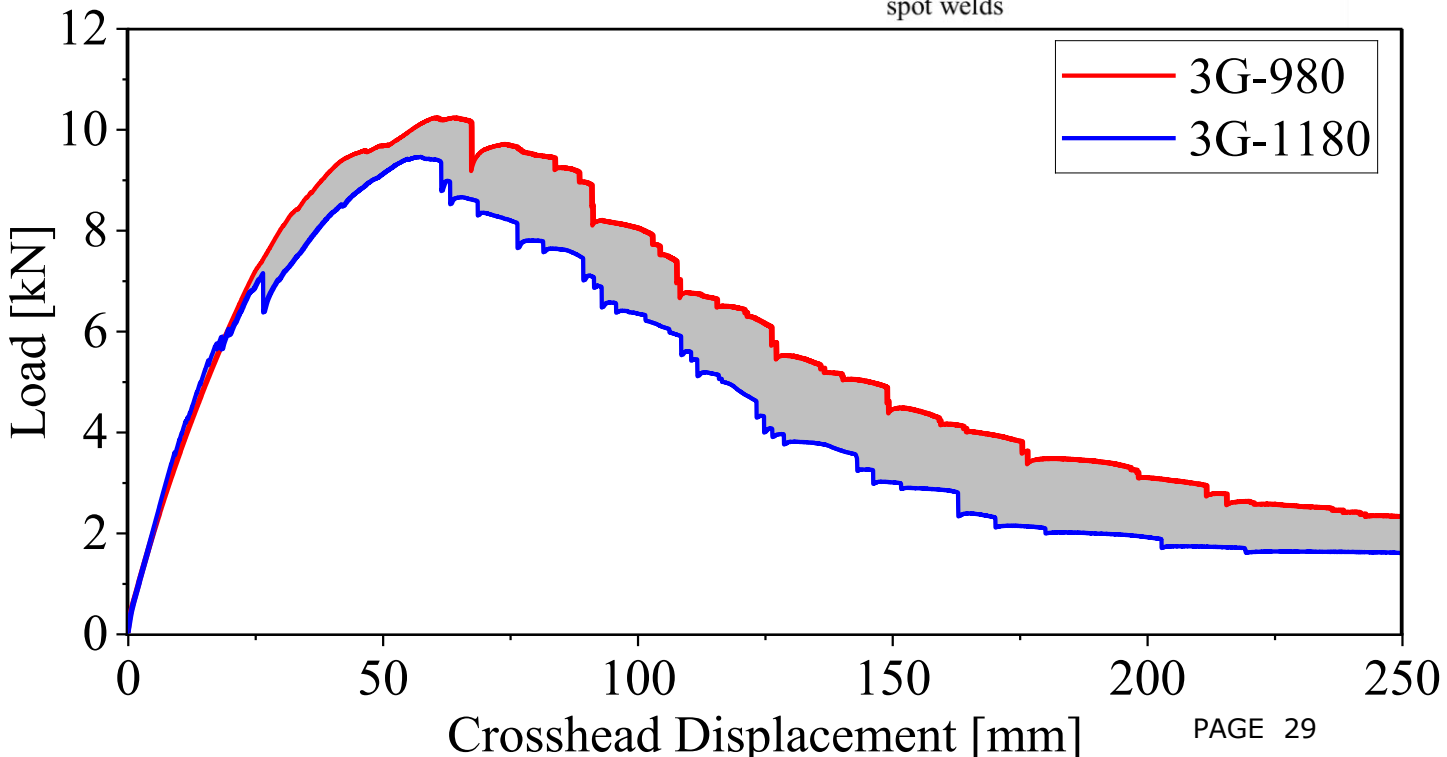
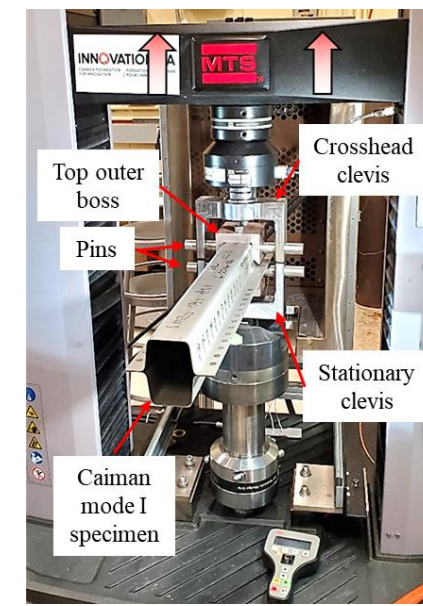
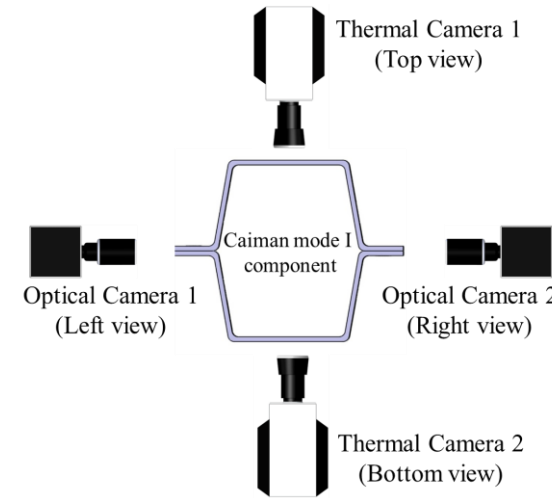
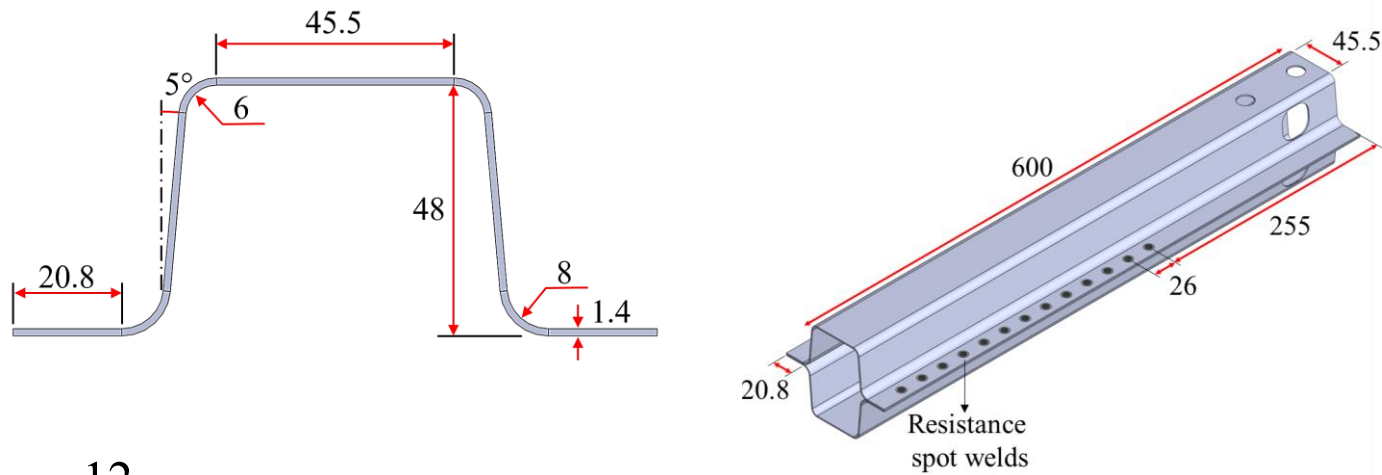
# Mechanical Performance and Failure Behavior of Spot Weld Groups Under Tensile-bending Loading Condition



Height  
[mm]



# RSW Weld-group tests methodology: The Caiman test



**3G-980 Components exhibited:**

- 13% higher energy absorption capability

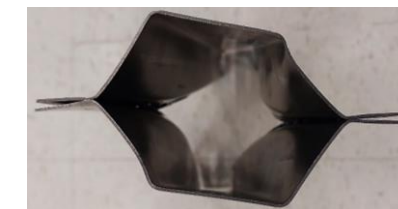
What are the primary factors influencing Energy absorption capability?

# Comparison of Parent Metal Deformation During Caiman Tests: 3G-980

3G-1180

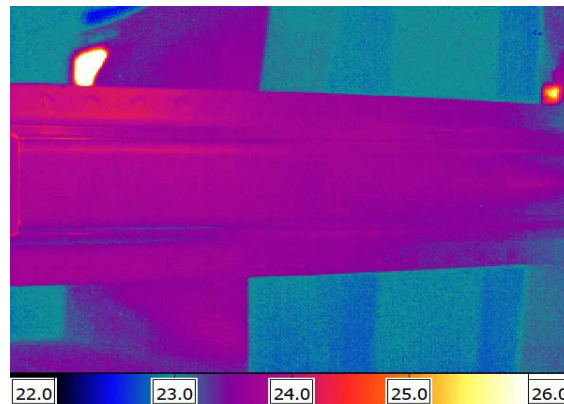
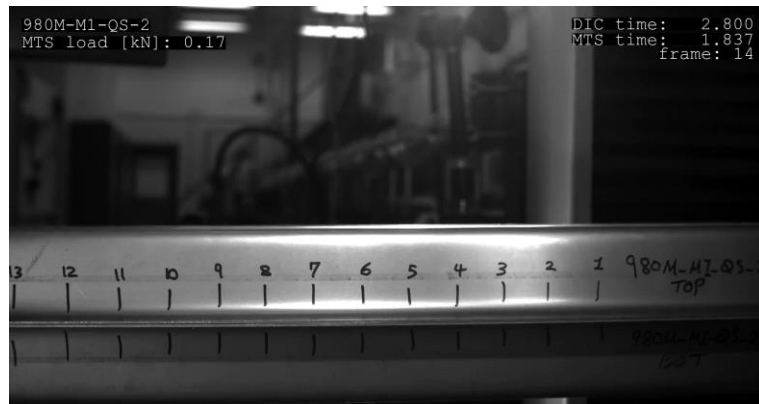
Optical sideview footage of Caiman tests:

Thermal footage of Caiman tests:

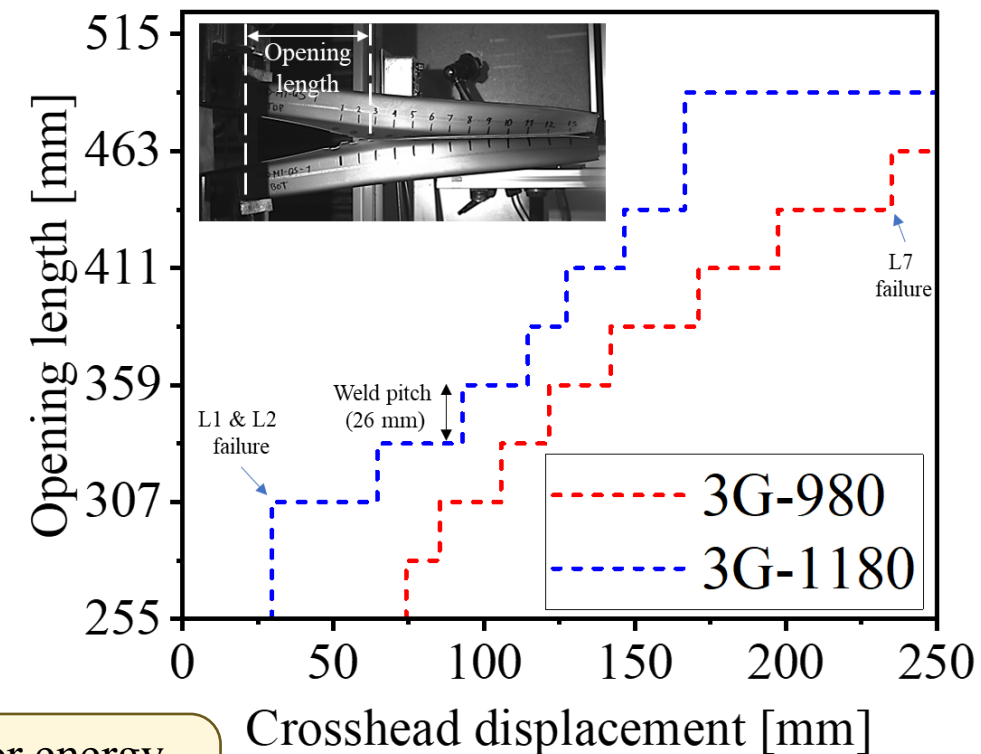
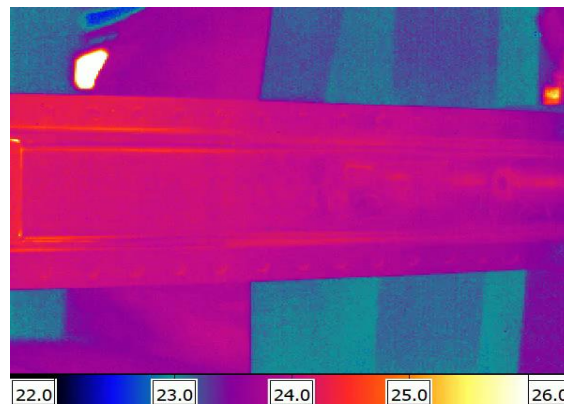


More severe sidewall collapse

3G-980



3G-1180



More severe collapse of the 3G-980 channels



Higher plastic work due to parent metal deformation



Superior energy absorption capability

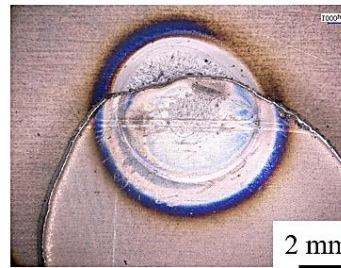
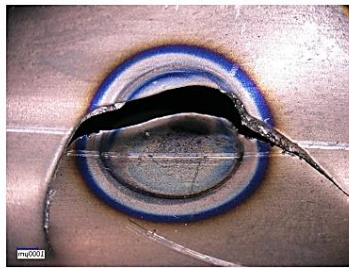
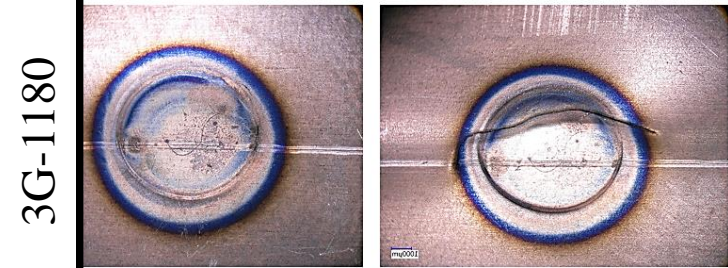
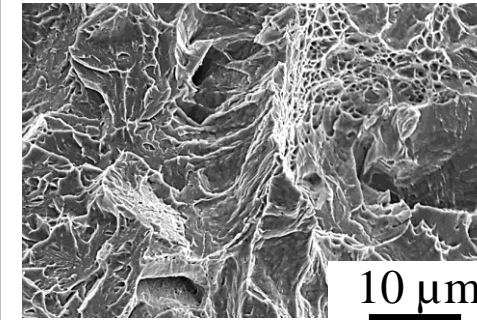
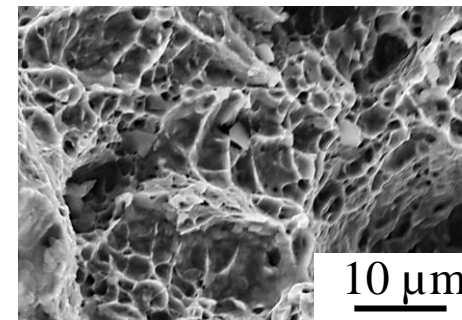
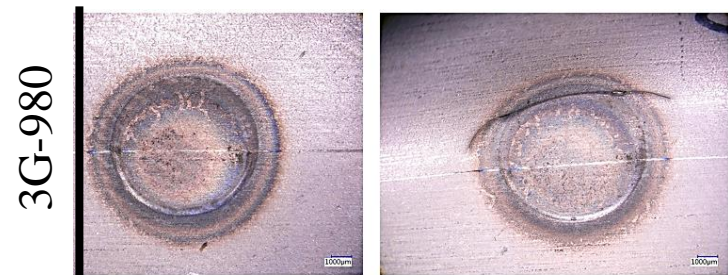
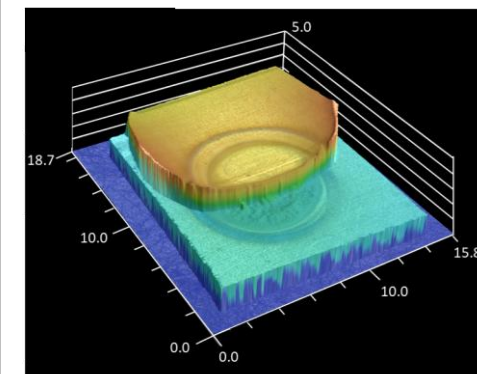
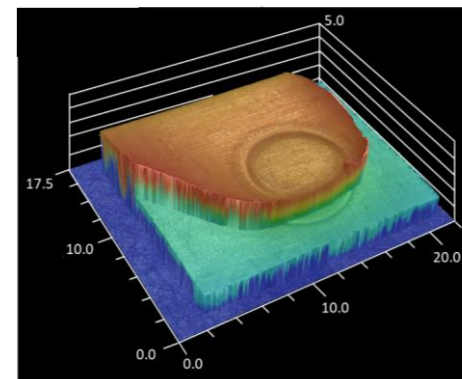
# Comparison of Failure Behaviors During Caiman Tests:

3G-980

3G-1180

3G-980

3G-1180



Dimple features

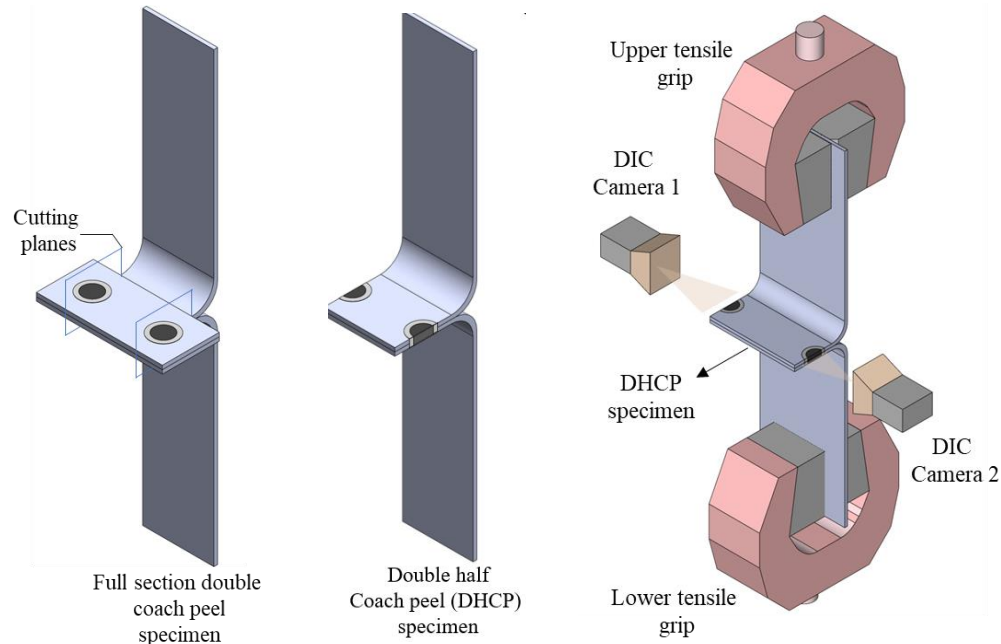
Dimple & Cleavage features

Ductile failure

Mixed failure

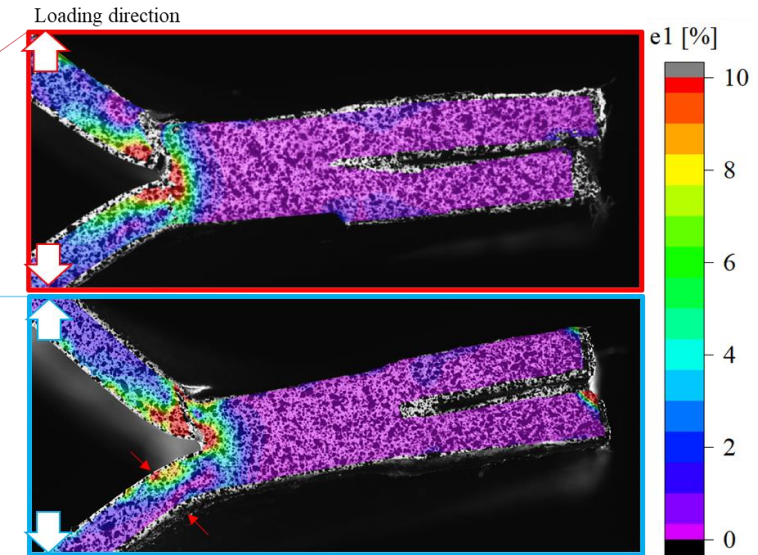
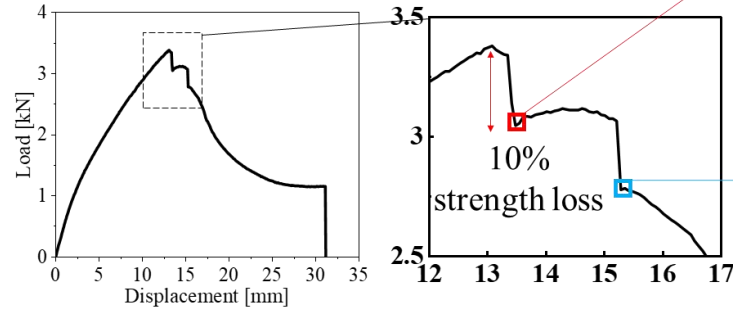
# Effect of Failure Mode on Mechanical Performance of RSW Groups

In-situ observation of sequential spot weld failure under tensile bending loads using modified coach peel test:

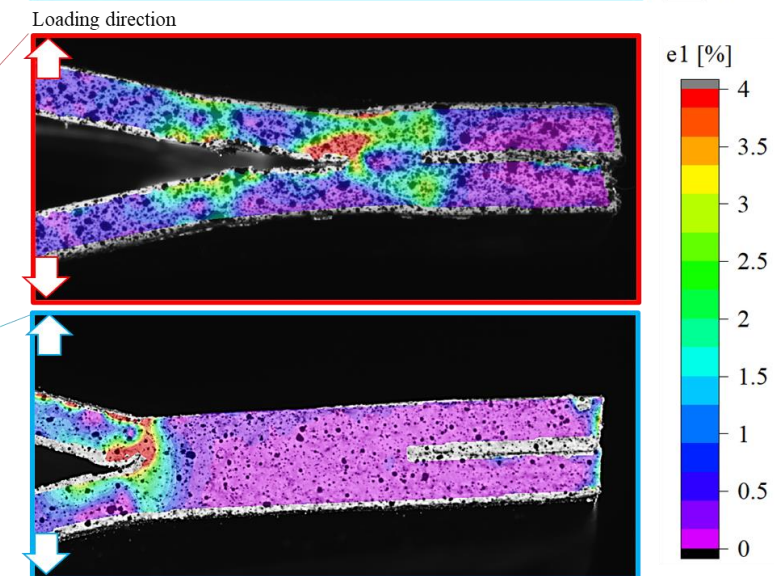
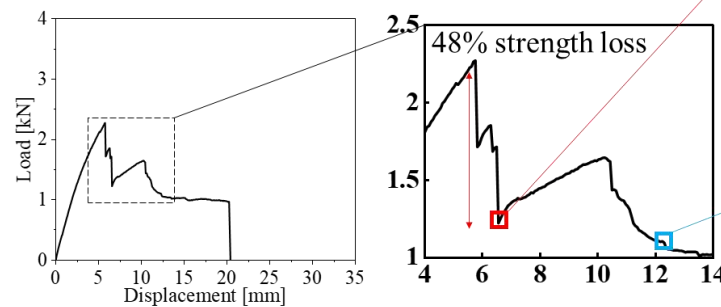


Brittle partial interfacial/ partial pullout failure mode leads to catastrophic loss of strength in sequential failure of spot welds

## 3G-980

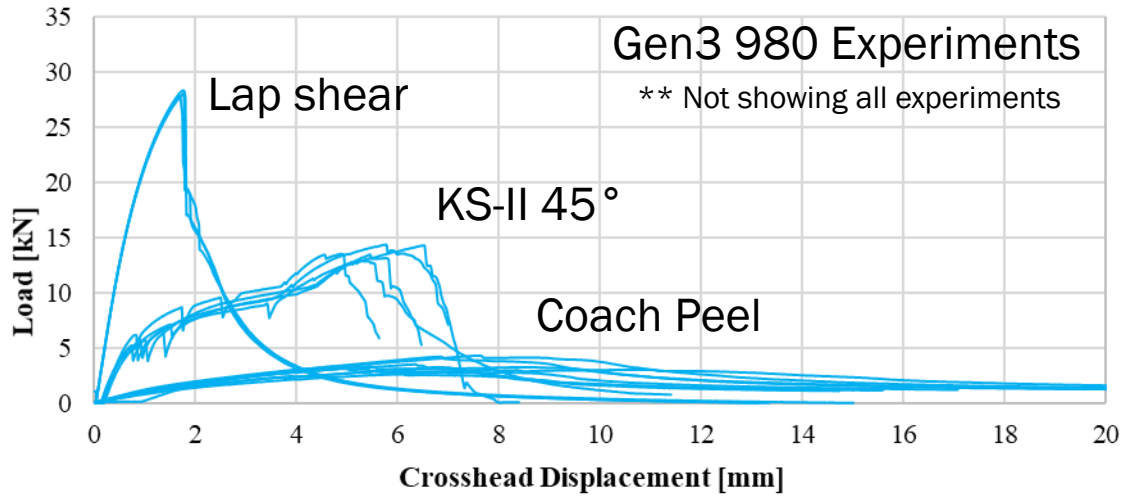


## 3G-1180

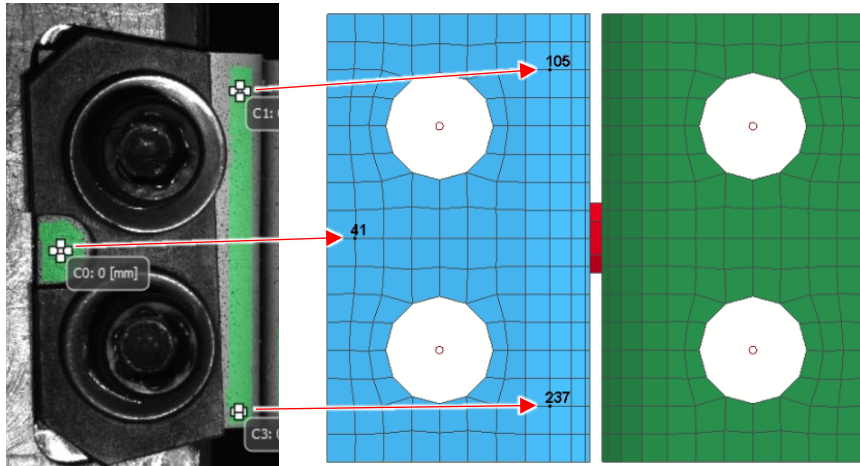


# Characterization and simulation of single spot weld

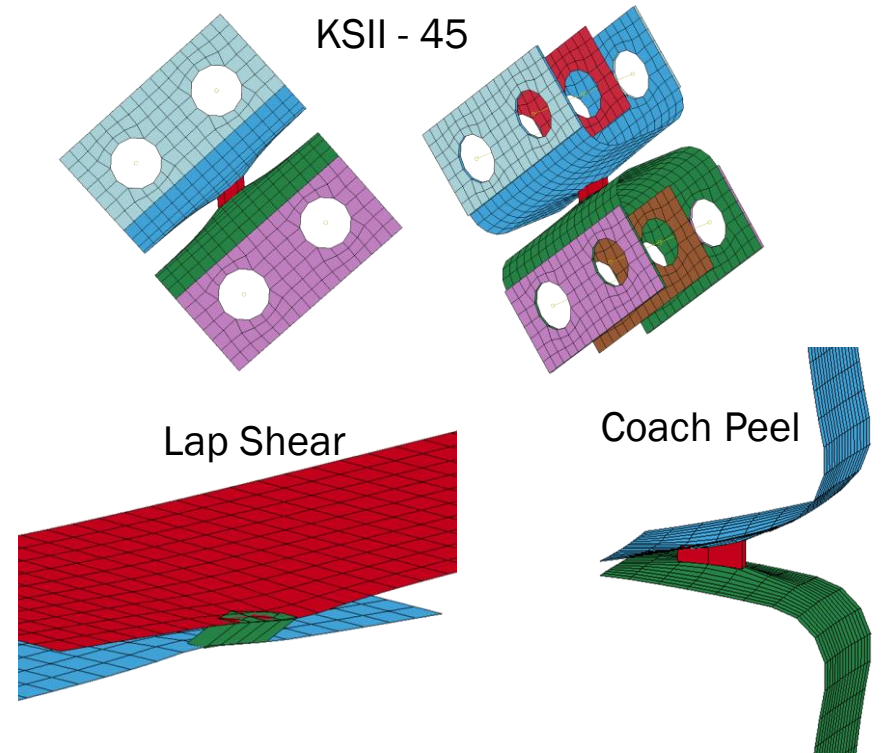
1. Process experimental data force vs displacement – target for simulations



2. Extract simulation boundary conditions from experiments  
- Novel method of KS-II 3D DIC analysis



3. Develop LS-DYNA simulation models of each single spot weld experiment – CAE-level, industry standard



Initial calculation for model bending parameter:

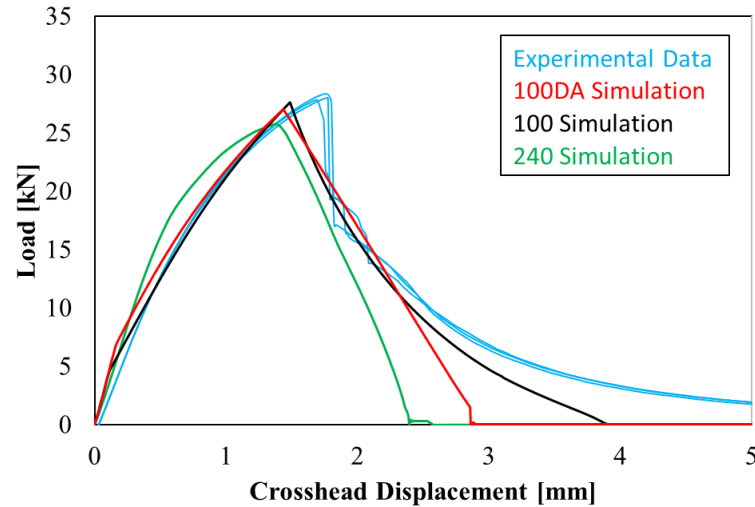
$$Bending\ moment\ (M)\ [kN.m] = \frac{F_{Max\ CP}\ [kN] \times L_{CP}\ [m]}{\sqrt{1 - \left(\frac{F_{Max\ CP}\ [kN]}{F_{Max\ KS-II\ 90^\circ}\ [kN]}\right)^2}}$$

\*\* Not showing all simulation models

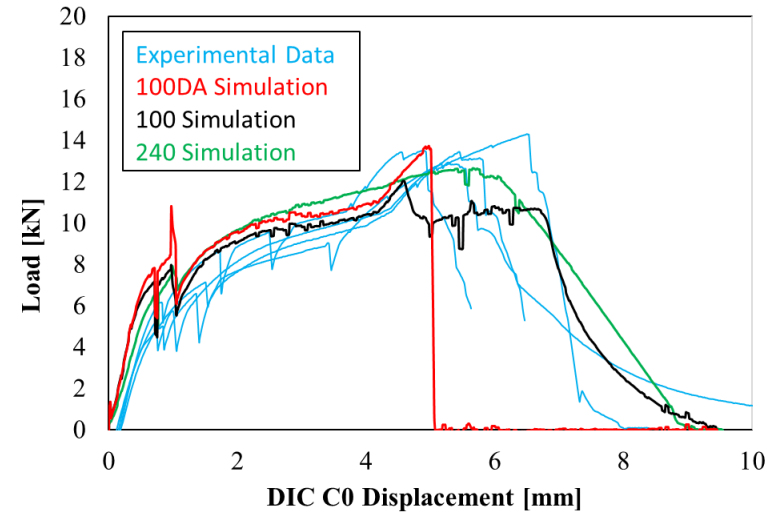
# Characterization and simulation of single spot weld

## 4. Calibrate three different LS-DYNA material models to compare performance

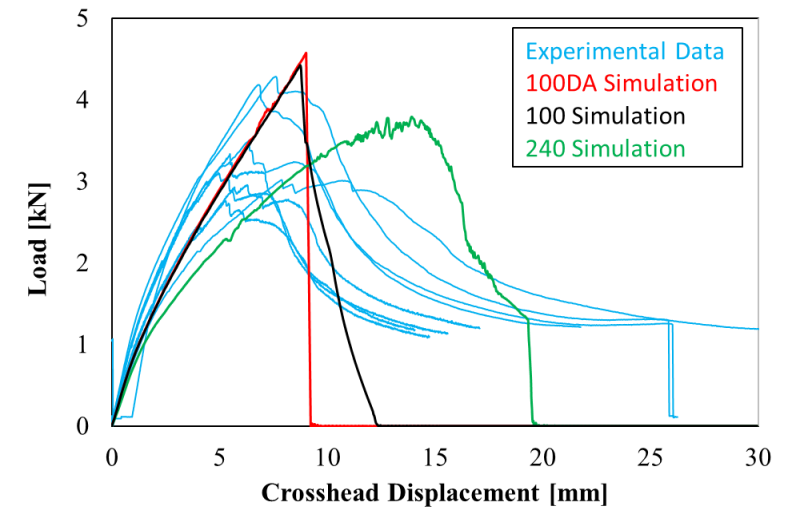
Gen3 980 - Lap Shear



Gen3 980 - KS-II 45



Gen3 980 - Coach Peel



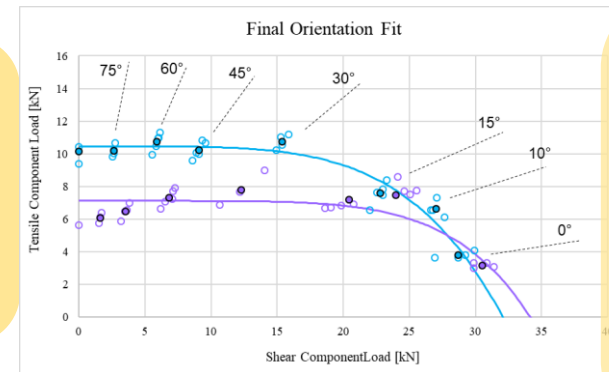
Three joint models examined, good agreement between models and experiments for tensile-shear and mixed loading conditions

### Current model failure criteria have difficulty accurately predicting coach peel

- MAT\_100: Has tensile, shear, and bending term in failure model, but limited fracture definitions
- MAT\_100DA: Similar to MAT\_100, but had to manually over predict bending load to predict component test
- MAD\_240: Good fracture behavior but no specific bending term in failure model

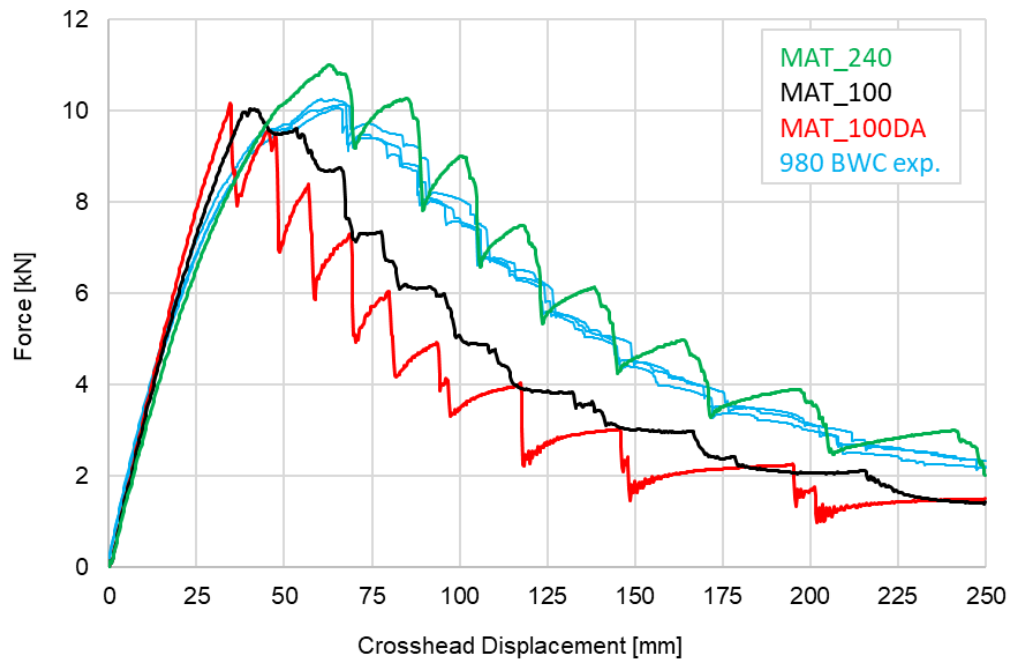
# Characterization and simulation of single spot weld

- Spot weld models calibrated from single spot weld experiments
- MAT\_240 does best, but all models have difficulty predicting component experiments
- Suspected due to limitations in model failure criterion

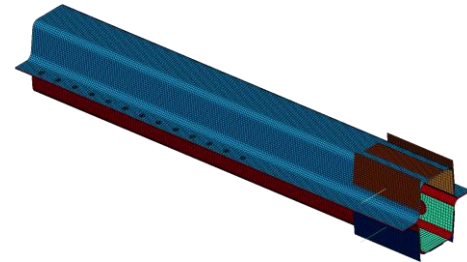
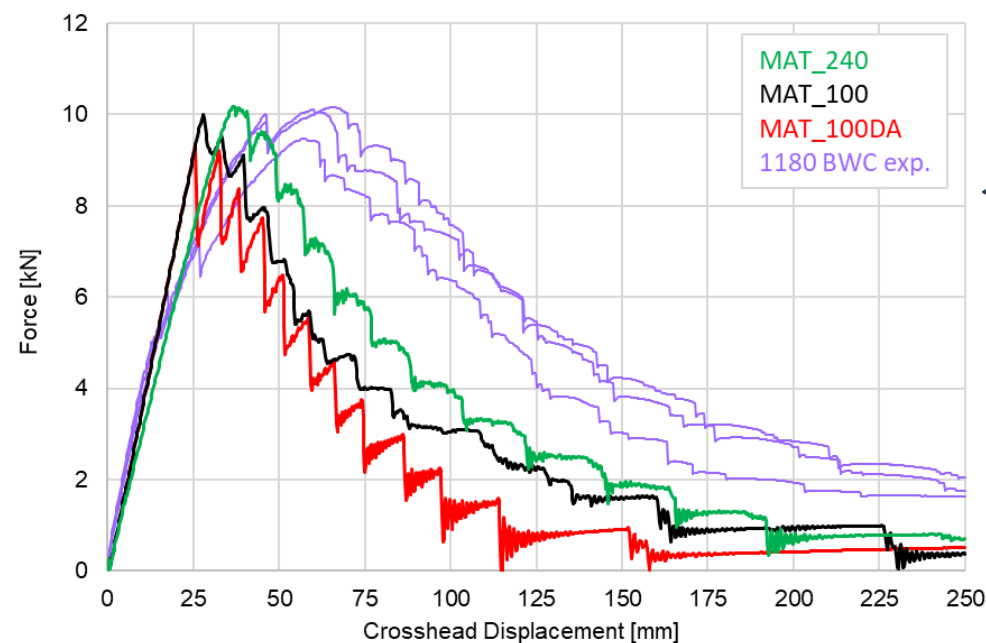


- Calibrated weld failure surface from KS-II single spot weld tests
- Test data has been corrected from 'ideal' fixture orientation to final coupon rotation at failure

### 3G-980 Caiman Mode 1 (quasi-static)



### 3G-1180 Caiman Mode 1 (quasi-static)



## Conclusions:

This study accounted for the **evolution of load components and moments** until the onset of joint failure during KS-II and CP tests, assessed the **accuracy of conventionally accepted force-based failure criteria**, and clarified the correlation between **mechanical performance and failure behavior of resistance spot weld-groups** under tensile-bending loading conditions.

- Seeger's criteria predicted the failure of 3G-AHSS spot welds in **shear-tension** loading accurately **(average over-prediction of 3.5%)**
- Seeger's criteria cannot be extended to **bending-dominated** loading conditions **(average over-prediction of joint bending strength of 66%)**
- Novel DIC-based methodologies were proposed that account for:
  - The evolution of geometry during CP tests and the forces/moments applied to the joint
  - The final ratio of shear/tensile forces during KS-II tests applied to the joint
- **Two factors** dominate the energy absorption capability of 3G-AHSS spot weld-group under tensile-bending loading:
  - **Plastic work** due to **parent metal** deformation
  - **Joint failure behavior**: Failure through the FZ will lead to catastrophic strength loss vs smooth unloading during PO
- The single spot weld mechanical properties were used to build industry-standard LS-DYNA material cards that predict the spot weld failure under combinations of load scenarios.

## Q & A:

Please use the Q & A feature to ask your questions



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